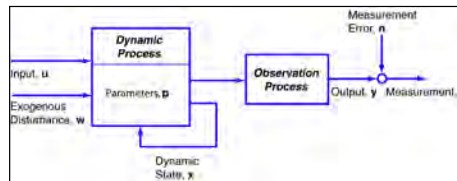


# Translational and Rotational Dynamics

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Robotics and Intelligent Systems MAE 345,  
Princeton University, 2017



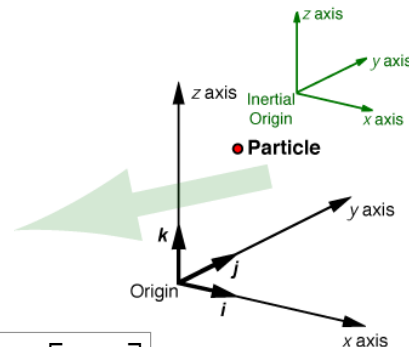
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<http://www.princeton.edu/~stengel/MAE345.html>

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## Reference Frame

- **Newtonian (Inertial) Frame of Reference**
  - **Unaccelerated Cartesian frame**
    - **Origin referenced to inertial (non-moving) frame**
  - **Right-hand rule**
  - **Origin can translate at constant linear velocity**
  - **Frame cannot rotate with respect to inertial origin**
- **Position: 3 dimensions**
  - **What is a non-moving frame?**



$$\mathbf{r} = \begin{bmatrix} x \\ y \\ z \end{bmatrix}$$

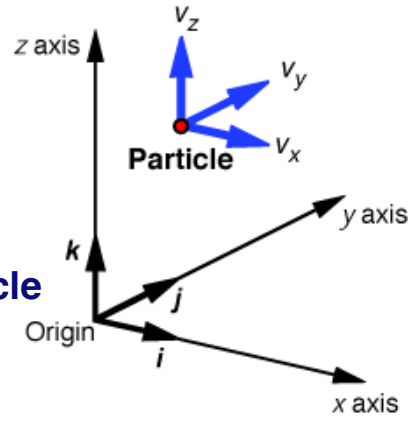
- **Translation = Linear motion**

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# Velocity and Momentum of a Particle

- **Velocity of a particle**

$$\mathbf{v} = \frac{d\mathbf{r}}{dt} = \dot{\mathbf{r}} = \begin{bmatrix} \dot{x} \\ \dot{y} \\ \dot{z} \end{bmatrix} = \begin{bmatrix} v_x \\ v_y \\ v_z \end{bmatrix}$$



- **Linear momentum of a particle**

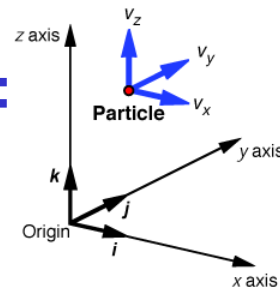
$$\mathbf{p} = m\mathbf{v} = m \begin{bmatrix} v_x \\ v_y \\ v_z \end{bmatrix}$$

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## Newton's Laws of Motion: Dynamics of a Particle

### First Law

- **If no force acts on a particle,**  
it remains at rest or continues to move in  
straight line at constant velocity,
- **Inertial reference frame**
- **Momentum is conserved**



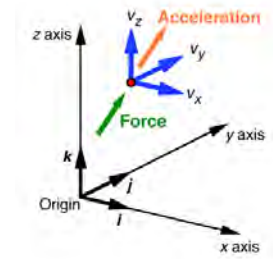
$$\frac{d}{dt}(m\mathbf{v}) = 0 \quad ; \quad m\mathbf{v}|_{t_1} = m\mathbf{v}|_{t_2}$$

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# Newton's Laws of Motion: Dynamics of a Particle

## Second Law

- Particle acted upon by force
- Acceleration proportional to and in direction of force
- Inertial reference frame
- Ratio of force to acceleration is particle mass



$$\frac{d}{dt}(m\mathbf{v}) = m \frac{d\mathbf{v}}{dt} = m\mathbf{a} = \mathbf{Force}$$

$$\mathbf{Force} = \begin{bmatrix} f_x \\ f_y \\ f_z \end{bmatrix} = \text{force vector}$$

$$\therefore \frac{d\mathbf{v}}{dt} = \frac{1}{m} \mathbf{Force} = \frac{1}{m} \mathbf{I}_3 \mathbf{Force}$$

$$= \begin{bmatrix} 1/m & 0 & 0 \\ 0 & 1/m & 0 \\ 0 & 0 & 1/m \end{bmatrix} \begin{bmatrix} f_x \\ f_y \\ f_z \end{bmatrix}$$

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# Newton's Laws of Motion: Dynamics of a Particle

## Third Law

For every **action**, there is an equal and opposite **reaction**



Force on rocket motor = -Force on exhaust gas

$$\mathbf{F}_R = -\mathbf{F}_E$$

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# One-Degree-of-Freedom Example of Newton's Second Law

2<sup>nd</sup>-order, linear, time-invariant ordinary differential equation

$$\frac{d^2 x(t)}{dt^2} \triangleq \ddot{x}(t) = \dot{v}_x(t) = \frac{f_x(t)}{m}$$

$\triangleq$  "Defined as"

Corresponding set of 1<sup>st</sup>-order equations  
(State-Space Model)

$$\begin{aligned} \frac{dx_1(t)}{dt} &\triangleq \dot{x}_1(t) \triangleq x_2(t) \triangleq v_x(t) \\ \frac{dx_2(t)}{dt} &\triangleq \dot{x}_2(t) = \dot{v}_x(t) = \frac{f_x(t)}{m} \end{aligned}$$

$$\begin{aligned} x_1(t) &\triangleq x(t), \text{ Displacement} \\ x_2(t) &\triangleq \frac{dx(t)}{dt}, \text{ Rate} \end{aligned}$$

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## State-Space Model is a Set of 1<sup>st</sup>-Order Ordinary Differential Equations

State, control, and output vectors for the example

$$\mathbf{x}(t) = \begin{bmatrix} x_1(t) \\ x_2(t) \end{bmatrix}; \quad \mathbf{u}(t) = u(t) = f_x(t); \quad \mathbf{y}(t) = \begin{bmatrix} x_1(t) \\ x_2(t) \end{bmatrix}$$

Stability and control-effect matrices

$$\mathbf{F} = \begin{bmatrix} 0 & 1 \\ 0 & 0 \end{bmatrix}; \quad \mathbf{G} = \begin{bmatrix} 0 \\ 1/m \end{bmatrix}$$

Dynamic equation

$$\dot{\mathbf{x}}(t) = \mathbf{F} \mathbf{x}(t) + \mathbf{G} \mathbf{u}(t)$$

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# State-Space Model of the 1-DOF Example

## Output equation

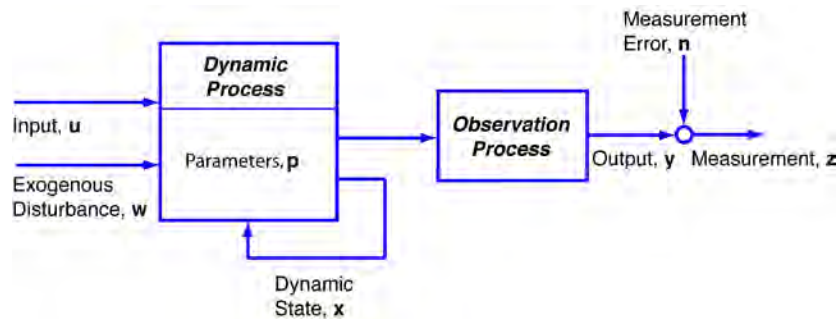
$$\mathbf{y}(t) = \mathbf{H}_x \mathbf{x}(t) + \mathbf{H}_u \mathbf{u}(t)$$

## Output coefficient matrices

$$\mathbf{H}_x = \begin{bmatrix} 1 & 0 \\ 0 & 1 \end{bmatrix}; \quad \mathbf{H}_u = \begin{bmatrix} 0 \\ 0 \end{bmatrix}$$

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## Dynamic System



**Dynamic Process: Current state may depend on prior state**

**x** : state  $dim = (n \times 1)$   
**u** : input  $dim = (m \times 1)$   
**w** : disturbance  $dim = (s \times 1)$   
**p** : parameter  $dim = (\ell \times 1)$

**t** : time (independent variable,  $1 \times 1$ )

**Observation Process: Measurement may contain error or be incomplete**

**y** : output (error-free)  $dim = (r \times 1)$   
**n** : measurement error  $dim = (r \times 1)$   
**z** : measurement  $dim = (r \times 1)$

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# State-Space Model of Three-Degree-of-Freedom Dynamics

$$\dot{\mathbf{x}}(t) \triangleq \begin{bmatrix} \dot{\mathbf{r}}(t) \\ \dot{\mathbf{v}}(t) \end{bmatrix} = \begin{bmatrix} \dot{x} \\ \dot{y} \\ \dot{z} \\ \dot{v}_x \\ \dot{v}_y \\ \dot{v}_z \end{bmatrix} = \begin{bmatrix} v_x \\ v_y \\ v_z \\ f_x/m \\ f_y/m \\ f_z/m \end{bmatrix} = \mathbf{F}\mathbf{x}(t) + \mathbf{G}\mathbf{u}(t)$$

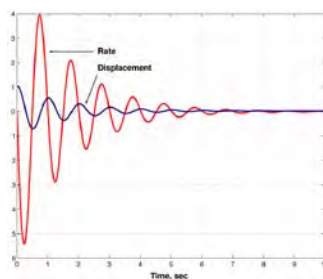
$$\mathbf{x}(t) \triangleq \begin{bmatrix} \mathbf{r}(t) \\ \mathbf{v}(t) \end{bmatrix} = \begin{bmatrix} x \\ y \\ z \\ v_x \\ v_y \\ v_z \end{bmatrix}$$

$$= \begin{bmatrix} 0 & 0 & 0 & 1 & 0 & 0 \\ 0 & 0 & 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 0 & 0 & 1 \\ 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 \end{bmatrix} \begin{bmatrix} x \\ y \\ z \\ v_x \\ v_y \\ v_z \end{bmatrix} + \begin{bmatrix} 0 & 0 & 0 \\ 0 & 0 & 0 \\ 0 & 0 & 0 \\ 1/m & 0 & 0 \\ 0 & 1/m & 0 \\ 0 & 0 & 1/m \end{bmatrix} \begin{bmatrix} f_x \\ f_y \\ f_z \end{bmatrix}$$

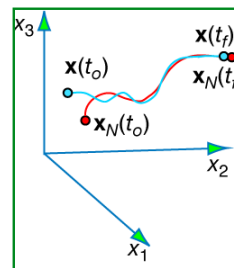
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## What Use are the Dynamic Equations?

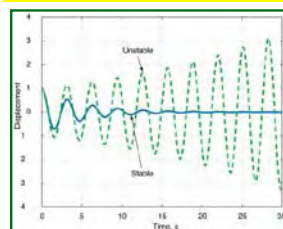
### Compute time response



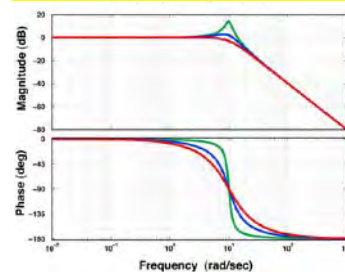
### Compute trajectories



### Determine stability



### Identify modes of motion



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# Forces

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## External Forces: Aerodynamic/Hydrodynamic

$$\mathbf{f}_{aero} = \begin{bmatrix} X \\ Y \\ Z \end{bmatrix} = \begin{bmatrix} C_X \\ C_Y \\ C_Z \end{bmatrix} \frac{1}{2} \rho V^2 A$$

$\rho$  = air density, function of height

$$= \rho_{sealevel} e^{-\beta h}$$

$V$  = airspeed

$$= [v_x^2 + v_y^2 + v_z^2]^{1/2} = [\mathbf{v}^T \mathbf{v}]^{1/2}$$

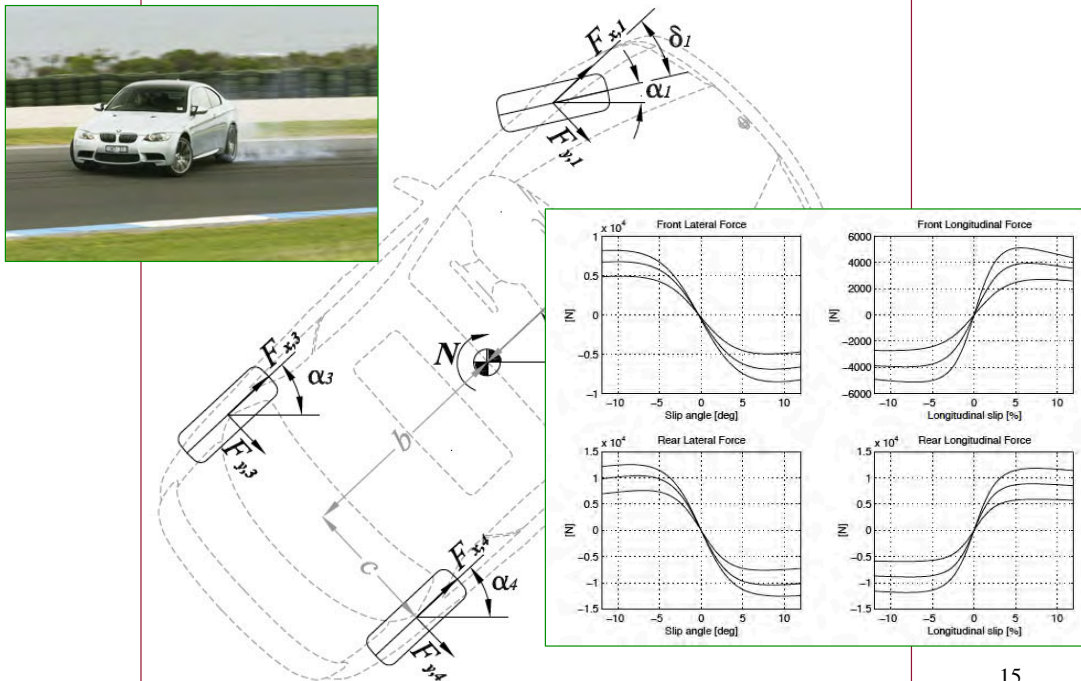
$A$  = reference area

$$\begin{bmatrix} C_X \\ C_Y \\ C_Z \end{bmatrix} = \begin{matrix} \text{dimensionless} \\ \text{aerodynamic coefficients} \end{matrix}$$

**Inertial frame or body frame?**

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## External Forces: Friction



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## External Forces: Gravity



- Flat-earth approximation
  - $g$  is gravitational acceleration
  - $mg$  is gravitational force
  - Independent of position
  - $z$  measured up

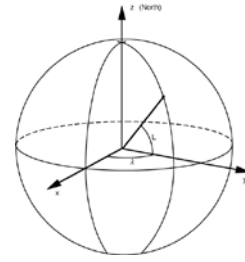
$$m\mathbf{g}_{flat} = m \begin{bmatrix} 0 \\ 0 \\ -g_o \end{bmatrix}; \quad g_o = 9.807 \text{ m/s}^2$$



# External Forces: Gravity

$$r = (x^2 + y^2 + z^2)^{1/2}$$

$L = \text{Latitude}$   
 $\lambda = \text{Longitude}$



- Spherical earth, inertial frame
  - "Inverse-square" gravitation
  - Non-linear function of position
  - $\mu = 3.986 \times 10^{14} \text{ m}^3/\text{s}^2$
- Spherical earth, rotating frame
  - "Inverse-square" gravitation
  - "Centripetal acceleration"
  - $\Omega = 7.29 \times 10^{-5} \text{ rad/s}$

$$\mathbf{g}_{\text{ground}} = \begin{bmatrix} g_x \\ g_y \\ g_z \end{bmatrix} = \mathbf{g}_{\text{gravity}}$$

$$= -\frac{\mu}{r^3} \begin{bmatrix} x \\ y \\ z \end{bmatrix} = -\frac{\mu}{r^2} \begin{bmatrix} x/r \\ y/r \\ z/r \end{bmatrix} = -\frac{\mu}{r^2} \begin{bmatrix} \cos L \cos \lambda \\ \cos L \sin \lambda \\ \sin L \end{bmatrix}$$

$$\mathbf{g}_r = \mathbf{g}_{\text{gravity}} + \mathbf{g}_{\text{rotation}}$$

$$= -\frac{\mu}{r^3} \begin{bmatrix} x \\ y \\ z \end{bmatrix} + \Omega^2 \begin{bmatrix} x \\ y \\ 0 \end{bmatrix}$$

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## State-Space Model with Round-Earth Gravity Model (Non-Rotating Frame)

$$\begin{bmatrix} \dot{x} \\ \dot{y} \\ \dot{z} \\ \dot{v}_x \\ \dot{v}_y \\ \dot{v}_z \end{bmatrix} = \begin{bmatrix} 0 & 0 & 0 & 1 & 0 & 0 \\ 0 & 0 & 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 0 & 0 & 1 \\ \hline 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 \end{bmatrix} \begin{bmatrix} x \\ y \\ z \\ v_x \\ v_y \\ v_z \end{bmatrix} - \begin{bmatrix} 0 & 0 & 0 \\ 0 & 0 & 0 \\ 0 & 0 & 0 \\ \hline \mu/r^3 & 0 & 0 \\ 0 & \mu/r^3 & 0 \\ 0 & 0 & \mu/r^3 \end{bmatrix} \begin{bmatrix} x \\ y \\ z \end{bmatrix}$$

$$= \begin{bmatrix} 0 & 0 & 0 & 1 & 0 & 0 \\ 0 & 0 & 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 0 & 0 & 1 \\ \hline -\mu/r^3 & 0 & 0 & 0 & 0 & 0 \\ 0 & -\mu/r^3 & 0 & 0 & 0 & 0 \\ 0 & 0 & -\mu/r^3 & 0 & 0 & 0 \end{bmatrix} \begin{bmatrix} x \\ y \\ z \\ v_x \\ v_y \\ v_z \end{bmatrix}$$

**Inverse-square gravity model introduces nonlinearity**

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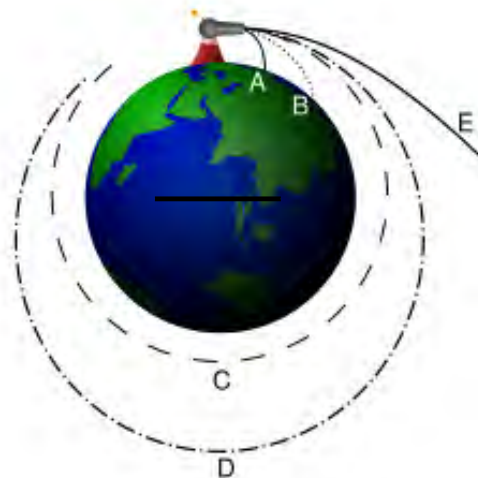
## Vector-Matrix Form of Round-Earth Dynamic Model

$$\begin{bmatrix} \dot{\mathbf{r}} \\ \dot{\mathbf{v}} \end{bmatrix} = \begin{bmatrix} \mathbf{0} & \mathbf{I}_3 \\ -\frac{\mu}{r^3} \mathbf{I}_3 & \mathbf{0} \end{bmatrix} \begin{bmatrix} \mathbf{r} \\ \mathbf{v} \end{bmatrix}$$

What other forces might be considered, and where would they appear in the model?

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## Point-Mass Motions of Spacecraft



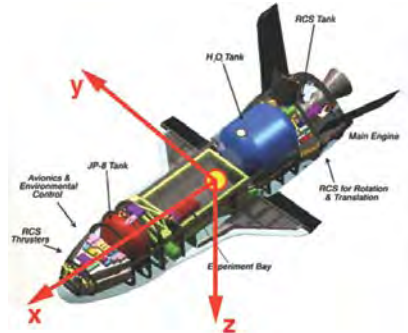
- For short distance and low speed, flat-Earth frame of reference and gravity are sufficient
- For long distance and high speed, round-Earth frame and inverse-square gravity are needed

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# Mass of an Object

$$m = \int_{\text{Body}} dm = \int_{x_{\min}}^{x_{\max}} \int_{y_{\min}}^{y_{\max}} \int_{z_{\min}}^{z_{\max}} \rho(x, y, z) dx dy dz$$

$\rho(x, y, z) = \text{density of the body}$



Density of object may vary with  $(x, y, z)$

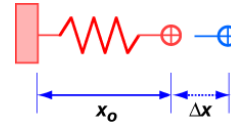
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## More Forces

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## External Forces: Linear Springs



### Scalar, linear spring

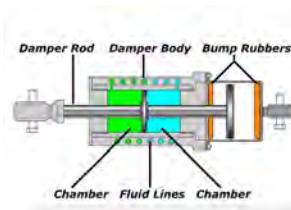
$$f = -k(x - x_o) = -k\Delta x; \quad k = \text{spring constant}$$

### Uncoupled, linear vector spring

$$\mathbf{f}_S = - \begin{bmatrix} k_x(x - x_o) \\ k_y(y - y_o) \\ k_z(z - z_o) \end{bmatrix} = - \begin{bmatrix} k_x & 0 & 0 \\ 0 & k_y & 0 \\ 0 & 0 & k_z \end{bmatrix} \begin{bmatrix} \Delta x \\ \Delta y \\ \Delta z \end{bmatrix}$$

$(x_o, y_o, z_o)$ : Equilibrium (reference) position

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## External Forces: Viscous Dampers

### Scalar, linear damper

$$f = -d(v - v_o) = -d\Delta v; \quad d = \text{damping constant}$$

### Uncoupled, linear vector damper

$$\mathbf{f}_D = - \begin{bmatrix} d_x(v_x - v_{x_o}) \\ d_y(v_y - v_{y_o}) \\ d_z(v_z - v_{z_o}) \end{bmatrix} \cong - \begin{bmatrix} d_x & 0 & 0 \\ 0 & d_y & 0 \\ 0 & 0 & d_z \end{bmatrix} \begin{bmatrix} v_x \\ v_y \\ v_z \end{bmatrix}$$

$(v_{x_o}, v_{y_o}, v_{z_o})$ : Equilibrium (reference) velocity [usually = 0]

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## State-Space Model with Linear Spring and Damping

$$\begin{bmatrix} \dot{x} \\ \dot{y} \\ \dot{z} \\ \dot{v}_x \\ \dot{v}_y \\ \dot{v}_z \end{bmatrix} = \begin{bmatrix} 0 & 0 & 0 & 1 & 0 & 0 \\ 0 & 0 & 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 0 & 0 & 1 \\ \hline 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 \end{bmatrix} \begin{bmatrix} x \\ y \\ z \\ v_x \\ v_y \\ v_z \end{bmatrix} + \begin{bmatrix} 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 \\ 0 & 0 & 0 & 0 & 0 & 0 \\ \hline \frac{-k_x}{m} & 0 & 0 & \frac{-d_x}{m} & 0 & 0 \\ 0 & \frac{-k_y}{m} & 0 & 0 & \frac{-d_y}{m} & 0 \\ 0 & 0 & \frac{-k_z}{m} & 0 & 0 & \frac{-d_z}{m} \end{bmatrix} \begin{bmatrix} (x-x_o) \\ (y-y_o) \\ (z-z_o) \\ v_x \\ v_y \\ v_z \end{bmatrix}$$

Spring Effects

Damping Effects

## State-Space Model with Linear Spring and Damping

Stability Effects

Reference Effects

$$\begin{bmatrix} \dot{x} \\ \dot{y} \\ \dot{z} \\ \dot{v}_x \\ \dot{v}_y \\ \dot{v}_z \end{bmatrix} = \begin{bmatrix} 0 & 0 & 0 & 1 & 0 & 0 \\ 0 & 0 & 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 0 & 0 & 1 \\ \hline \frac{-k_x}{m} & 0 & 0 & \frac{-d_x}{m} & 0 & 0 \\ 0 & \frac{-k_y}{m} & 0 & 0 & \frac{-d_y}{m} & 0 \\ 0 & 0 & \frac{-k_z}{m} & 0 & 0 & \frac{-d_z}{m} \end{bmatrix} \begin{bmatrix} x \\ y \\ z \\ v_x \\ v_y \\ v_z \end{bmatrix} + \begin{bmatrix} 0 & 0 & 0 & 1 & 0 & 0 \\ 0 & 0 & 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & 0 & 0 & 1 \\ \hline \frac{k_x}{m} & 0 & 0 & \frac{d_x}{m} & 0 & 0 \\ 0 & \frac{k_y}{m} & 0 & 0 & \frac{d_y}{m} & 0 \\ 0 & 0 & \frac{k_z}{m} & 0 & 0 & \frac{d_z}{m} \end{bmatrix} \begin{bmatrix} x_o \\ y_o \\ z_o \\ 0 \\ 0 \\ 0 \end{bmatrix}$$

## Vector-Matrix Form

$$\begin{bmatrix} \dot{\mathbf{r}} \\ \dot{\mathbf{v}} \end{bmatrix} = \begin{bmatrix} \mathbf{0} & \mathbf{I}_3 \\ -\mathbf{K}/m & -\mathbf{D}/m \end{bmatrix} \begin{bmatrix} (\mathbf{r} - \mathbf{r}_o) \\ \mathbf{v} \end{bmatrix}$$

$$\begin{bmatrix} \dot{\mathbf{r}} \\ \dot{\mathbf{v}} \end{bmatrix} = \begin{bmatrix} \mathbf{0} & \mathbf{I}_3 \\ -\mathbf{K}/m & -\mathbf{D}/m \end{bmatrix} \begin{bmatrix} \mathbf{r} \\ \mathbf{v} \end{bmatrix} + \begin{bmatrix} \mathbf{0} & -\mathbf{I}_3 \\ \mathbf{K}/m & \mathbf{D}/m \end{bmatrix} \begin{bmatrix} \mathbf{r}_o \\ \mathbf{0} \end{bmatrix}$$

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## *Rotational Motion*



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## Assignment # 2

Document the physical characteristics and flight behavior of a *Syma X11 quadcopter*

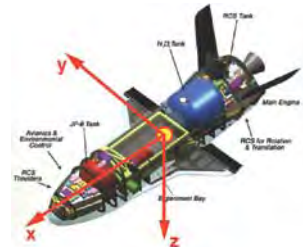


<https://www.youtube.com/watch?v=EwO6U7DbqSo>

<https://www.youtube.com/watch?v=kyiuy2CHzj0>

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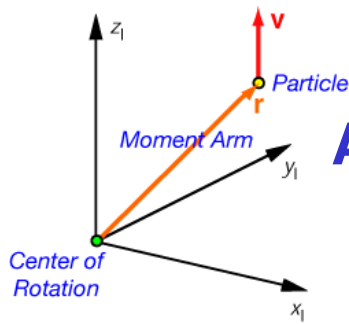
### Center of Mass



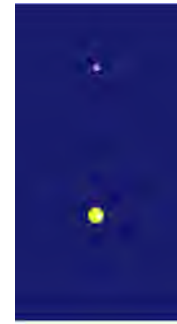
$$\mathbf{r}_{cm} \triangleq \begin{bmatrix} x_{cm} \\ y_{cm} \\ z_{cm} \end{bmatrix} = \frac{1}{m} \int_{Body} \mathbf{r} dm$$
$$= \int_{x_{min}}^{x_{max}} \int_{y_{min}}^{y_{max}} \int_{z_{min}}^{z_{max}} \begin{bmatrix} x \\ y \\ z \end{bmatrix} \rho(x, y, z) dx dy dz$$

Reference point for rotational motion

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Particle in Inverse-Square Field



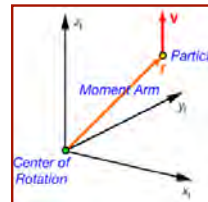
## Angular Momentum of a Particle

- **Moment of linear momentum** of differential particles that make up the body
  - Differential mass of a particle **times**
  - Component of velocity **perpendicular to moment arm** from center of rotation to particle

$$d\mathbf{h} = (\mathbf{r} \times d\mathbf{m}\mathbf{v}) = (\mathbf{r} \times \mathbf{v})dm$$

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## Cross Product of Two Vectors



$$\mathbf{r} \times \mathbf{v} = \begin{vmatrix} \mathbf{i} & \mathbf{j} & \mathbf{k} \\ x & y & z \\ v_x & v_y & v_z \end{vmatrix} = (yv_z - zv_y)\mathbf{i} + (zv_x - xv_z)\mathbf{j} + (xv_y - yv_x)\mathbf{k}$$

$(\mathbf{i}, \mathbf{j}, \mathbf{k})$ : Unit vectors along  $(x, y, z)$

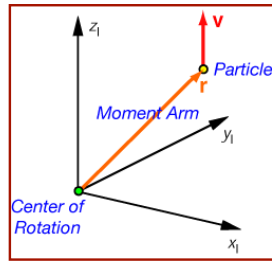
This is equivalent to

$$= \begin{bmatrix} (yv_z - zv_y) \\ (zv_x - xv_z) \\ (xv_y - yv_x) \end{bmatrix} = \begin{bmatrix} 0 & -z & y \\ z & 0 & -x \\ -y & x & 0 \end{bmatrix} \begin{bmatrix} v_x \\ v_y \\ v_z \end{bmatrix} = \tilde{\mathbf{r}}\mathbf{v}$$

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# Cross-Product-Equivalent Matrix



$$\mathbf{r} \times \mathbf{v} = \tilde{\mathbf{r}}\mathbf{v}$$

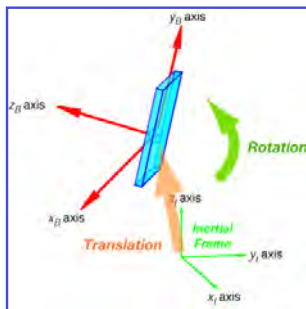
Cross-product equivalent of radius vector

$$\mathbf{r} \times \triangleq \tilde{\mathbf{r}} = \begin{bmatrix} 0 & -z & y \\ z & 0 & -x \\ -y & x & 0 \end{bmatrix}$$

Velocity vector

$$\mathbf{v} = \begin{bmatrix} v_x \\ v_y \\ v_z \end{bmatrix}$$

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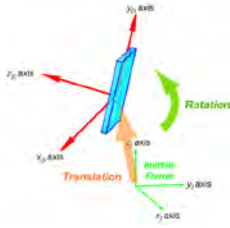
## Angular Momentum of an Object

Integrate moment of linear momentum of differential particles over the body

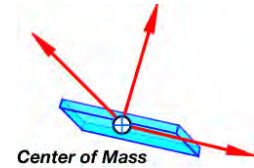
$$\mathbf{h} \triangleq \int_{\text{Body}} (\mathbf{r} \times \mathbf{v}) dm = \int_{x_{\min}}^{x_{\max}} \int_{y_{\min}}^{y_{\max}} \int_{z_{\min}}^{z_{\max}} (\mathbf{r} \times \mathbf{v}) \rho(x, y, z) dx dy dz$$

$$= \int_{x_{\min}}^{x_{\max}} \int_{y_{\min}}^{y_{\max}} \int_{z_{\min}}^{z_{\max}} (\tilde{\mathbf{r}}\mathbf{v}) \rho(x, y, z) dx dy dz \triangleq \begin{bmatrix} h_x \\ h_y \\ h_z \end{bmatrix}$$

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# Angular Velocity and Corresponding Velocity Increment



Angular velocity of object with respect to inertial frame of reference

$$\boldsymbol{\omega} = \begin{bmatrix} \omega_x \\ \omega_y \\ \omega_z \end{bmatrix}_I$$

Linear velocity increment at a point,  $(x, y, z)$ , due to angular rotation

$$\Delta \mathbf{v}(x, y, z) = \begin{bmatrix} \Delta v_x \\ \Delta v_y \\ \Delta v_z \end{bmatrix} = \boldsymbol{\omega} \times \mathbf{r} = -(\mathbf{r} \times \boldsymbol{\omega})$$

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## Angular Momentum of an Object with Respect to Its Center of Mass

- **Choose center of mass as origin about which angular momentum is calculated (= center of rotation)**
  - i.e.,  $\mathbf{r}$  is measured from the center of mass

$$\mathbf{h} = \int_{Body} [\mathbf{r} \times (\mathbf{v}_{cm} + \Delta \mathbf{v})] dm = \int_{Body} [\mathbf{r} \times \mathbf{v}_{cm}] dm + \int_{Body} [\mathbf{r} \times \Delta \mathbf{v}] dm$$

$$= \int_{Body} [\mathbf{r} dm] \times \mathbf{v}_{cm} - \int_{Body} [\mathbf{r} \times (\mathbf{r} \times \boldsymbol{\omega})] dm$$

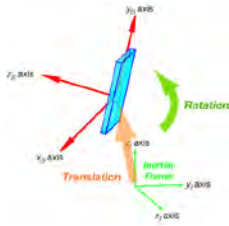
By symmetry,

$$\int_{Body} [\mathbf{r} dm] \times \mathbf{v}_{cm} = \mathbf{r}_{cm} \times \mathbf{v}_{cm} = 0$$

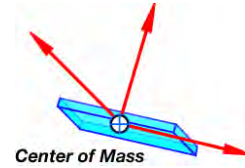
$$= - \int_{Body} \tilde{\mathbf{r}} \tilde{\mathbf{r}} \boldsymbol{\omega} dm = \left[ - \int_{Body} \tilde{\mathbf{r}} \tilde{\mathbf{r}} dm \right] \boldsymbol{\omega} \triangleq \mathbb{I} \boldsymbol{\omega}$$

$\mathbb{I} \triangleq$  Inertia Matrix

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# Angular Momentum



$$\mathbf{h} = \mathbb{I}\boldsymbol{\omega}$$

## Three components of angular momentum

$$\begin{bmatrix} h_x \\ h_y \\ h_z \end{bmatrix} = \left[ - \int_{Body} \tilde{\mathbf{r}} \tilde{\mathbf{r}} dm \right] \boldsymbol{\omega} \triangleq \begin{bmatrix} I_{xx} & -I_{xy} & -I_{xz} \\ -I_{xy} & I_{yy} & -I_{yz} \\ -I_{xz} & -I_{yz} & I_{zz} \end{bmatrix} \begin{bmatrix} \omega_x \\ \omega_y \\ \omega_z \end{bmatrix}$$

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## Inertia Matrix, $\mathbb{I}$

Angular rate has equal effect on all particles

$$\mathbb{I} = - \int_{Body} \tilde{\mathbf{r}} \tilde{\mathbf{r}} dm = - \int_{Body} \begin{bmatrix} 0 & -z & y \\ z & 0 & -x \\ -y & x & 0 \end{bmatrix} \begin{bmatrix} 0 & -z & y \\ z & 0 & -x \\ -y & x & 0 \end{bmatrix} dm$$

$$= \int_{Body} \begin{bmatrix} (y^2 + z^2) & -xy & -xz \\ -xy & (x^2 + z^2) & -yz \\ -xz & -yz & (x^2 + y^2) \end{bmatrix} dm$$

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# Inertia Matrix

$$\mathbb{I} = \int_{Body} \begin{bmatrix} (y^2 + z^2) & -xy & -xz \\ -xy & (x^2 + z^2) & -yz \\ -xz & -yz & (x^2 + y^2) \end{bmatrix} dm = \begin{bmatrix} I_{xx} & -I_{xy} & -I_{xz} \\ -I_{xy} & I_{yy} & -I_{yz} \\ -I_{xz} & -I_{yz} & I_{zz} \end{bmatrix}$$

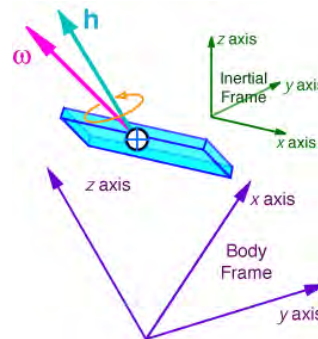
- **Moments of inertia** on the diagonal
- **Products of inertia** off the diagonal
- If **products of inertia are zero**, (x, y, z) are **principal axes**, and

$$\mathbb{I} = \begin{bmatrix} I_{xx} & 0 & 0 \\ 0 & I_{yy} & 0 \\ 0 & 0 & I_{zz} \end{bmatrix}$$

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## Angular Momentum and Rate are **Vectors**

- Can be expressed in either an **inertial** or a **body frame**
- Vectors are transformed by the **rotation matrix** and its **inverse**



$$\begin{aligned} \mathbf{h}_B &= \mathbb{I}_B \boldsymbol{\omega}_B = \mathbf{H}_I^B \mathbf{h}_I = \mathbf{H}_I^B \mathbb{I}_I \boldsymbol{\omega}_I \\ \boldsymbol{\omega}_B &= \mathbf{H}_I^B \boldsymbol{\omega}_I \end{aligned}$$

$$\begin{aligned} \mathbf{h}_I &= \mathbb{I}_I \boldsymbol{\omega}_I = \mathbf{H}_B^I \mathbf{h}_B = \mathbf{H}_B^I \mathbb{I}_B \boldsymbol{\omega}_B \\ \boldsymbol{\omega}_I &= \mathbf{H}_B^I \boldsymbol{\omega}_B \end{aligned}$$

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# Similarity Transformations for Matrices

Alternative expressions for angular momentum

$$\begin{aligned} \mathbf{h}_B &= \mathbb{I}_B \boldsymbol{\omega}_B = \mathbf{H}_I^B \mathbf{h}_I = \mathbf{H}_I^B \mathbb{I}_I \boldsymbol{\omega}_I \\ &= \mathbf{H}_I^B \mathbb{I}_I \mathbf{H}_B^I \boldsymbol{\omega}_B \end{aligned}$$

Inertia matrices are transformed from one frame to the other by **similarity transformations**

$$\mathbb{I}_B = \mathbf{H}_I^B \mathbb{I}_I \mathbf{H}_B^I$$

$$\mathbb{I}_I = \mathbf{H}_B^I \mathbb{I}_B \mathbf{H}_I^B$$

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## Newton's 2<sup>nd</sup> Law Applied to Rotational Motion in Inertial Frame

Rate of change of angular momentum  
= **applied moment (or torque),  $\mathbf{m}$**

Inertia Matrix Expressed in Inertial Frame is Not Constant if Body is Rotating

$$\frac{d\mathbf{h}_I}{dt} = \frac{d(\mathbb{I}_I \boldsymbol{\omega}_I)}{dt} = \frac{d\mathbb{I}_I}{dt} \boldsymbol{\omega}_I + \mathbb{I}_I \frac{d\boldsymbol{\omega}_I}{dt} = \mathbf{m}_I \quad [\text{moment vector}]_I$$

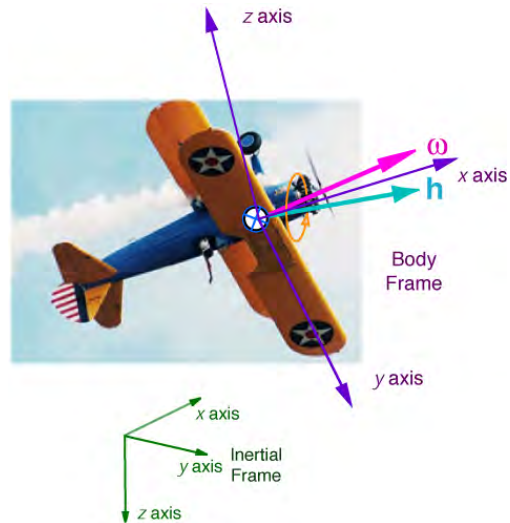
In a body reference frame, inertia matrix is constant

$$\begin{aligned} \frac{d\mathbf{h}_B}{dt} &= \frac{d(\mathbb{I}_B \boldsymbol{\omega}_B)}{dt} = \frac{d\mathbb{I}_B}{dt} \boldsymbol{\omega}_B + \mathbb{I}_B \frac{d\boldsymbol{\omega}_B}{dt} = (\mathbf{0}) \boldsymbol{\omega}_B + \mathbb{I}_B \frac{d\boldsymbol{\omega}_B}{dt} \\ &= \mathbb{I}_B \frac{d\boldsymbol{\omega}_B}{dt} = \mathbf{m}_B \quad [\text{moment vector}]_B \end{aligned}$$

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# How do We get Rid of $d(\mathbb{I}_I)/dt$ in the Angular Momentum Rate Equation?

- Write the dynamic equation in **body-referenced frame**
  - With constant mass, inertial properties are **unchanging** in body reference frame
  - ... but the frame is “non-Newtonian” or “non-inertial”



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## Vector Derivative Expressed in a Rotating Frame

Chain Rule

$$\frac{d\mathbf{h}_I}{dt} = \mathbf{H}_B^I \frac{d\mathbf{h}_B}{dt} + \frac{d\mathbf{H}_B^I}{dt} \mathbf{h}_B$$

$$= \mathbf{H}_B^I \frac{d\mathbf{h}_B}{dt} + (\boldsymbol{\omega}_I \times \mathbf{H}_B^I) \mathbf{h}_B = \mathbf{H}_B^I \frac{d\mathbf{h}_B}{dt} + \boldsymbol{\omega}_I \times (\mathbf{H}_B^I \mathbf{h}_B)$$

$$\frac{d\mathbf{h}_I}{dt} = \mathbf{H}_B^I \frac{d\mathbf{h}_B}{dt} + \boldsymbol{\omega}_I \times \mathbf{h}_I = \mathbf{H}_B^I \frac{d\mathbf{h}_B}{dt} + \tilde{\boldsymbol{\omega}}_I \mathbf{h}_I$$

Cross-product-equivalent matrix of angular rate:

$$\tilde{\boldsymbol{\omega}} = \begin{bmatrix} 0 & -\omega_z & \omega_y \\ \omega_z & 0 & -\omega_x \\ -\omega_y & \omega_x & 0 \end{bmatrix}$$

Similarity transformation

$$\tilde{\boldsymbol{\omega}}_I = \mathbf{H}_B^I \tilde{\boldsymbol{\omega}}_B \mathbf{H}_I^B$$

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# Rate of Change of Body-Axis Angular Momentum

Substitute

$$\frac{d\mathbf{h}_I}{dt} = \mathbf{H}_B^I \frac{d\mathbf{h}_B}{dt} + \mathbf{H}_B^I \tilde{\boldsymbol{\omega}}_B \mathbf{H}_I^B \mathbf{h}_I = \mathbf{m}_I = \mathbf{H}_B^I \mathbf{m}_B$$

Eliminate

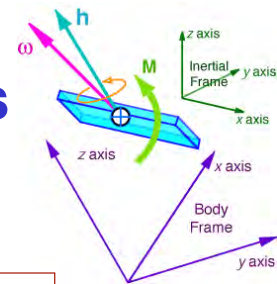
$$\mathbf{H}_B^I \frac{d\mathbf{h}_B}{dt} + \mathbf{H}_B^I \tilde{\boldsymbol{\omega}}_B \mathbf{H}_I^B \mathbf{h}_I = \mathbf{H}_B^I \mathbf{m}_B$$

Rearrange and Substitute

$$\frac{d\mathbf{h}_B}{dt} = \mathbf{m}_B - \tilde{\boldsymbol{\omega}}_B \mathbf{h}_B$$

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# Rate of Change of Body-Axis Angular Velocity



$$\mathbb{I}_B \frac{d\boldsymbol{\omega}_B}{dt} = \mathbf{m}_B - \tilde{\boldsymbol{\omega}}_B \mathbb{I}_B \boldsymbol{\omega}_B$$

$$\frac{d\boldsymbol{\omega}_B}{dt} = \mathbb{I}_B^{-1} (\mathbf{m}_B - \tilde{\boldsymbol{\omega}}_B \mathbb{I}_B \boldsymbol{\omega}_B)$$

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# Rate of Change of Body-Axis Translational Velocity

Inertial Velocity

$$\mathbf{v}_I = \mathbf{H}_B^I \mathbf{v}_B$$

Rate of change

$$\begin{aligned} \frac{d\mathbf{v}_I}{dt} &= \mathbf{H}_B^I \frac{d\mathbf{v}_B}{dt} + \frac{d(\mathbf{H}_B^I)}{dt} \mathbf{v}_B = \mathbf{H}_B^I \frac{d\mathbf{v}_B}{dt} + \tilde{\boldsymbol{\omega}}_I \mathbf{v}_I \\ &= \mathbf{H}_B^I \frac{d\mathbf{v}_B}{dt} + \mathbf{H}_B^I \tilde{\boldsymbol{\omega}}_B \mathbf{H}_B^I \mathbf{v}_I = \mathbf{H}_B^I \frac{d\mathbf{v}_B}{dt} + \mathbf{H}_B^I \tilde{\boldsymbol{\omega}}_B \mathbf{v}_B \\ &= \frac{1}{m} \mathbf{f}_I = \frac{1}{m} \mathbf{H}_B^I \mathbf{f}_B \end{aligned}$$

Substitution

$$\mathbf{H}_B^I \frac{d\mathbf{v}_B}{dt} + \mathbf{H}_B^I \tilde{\boldsymbol{\omega}}_B \mathbf{v}_B = \frac{1}{m} \mathbf{H}_B^I \mathbf{f}_B$$

Result

$$\frac{d\mathbf{v}_B}{dt} = \frac{1}{m} \mathbf{f}_B - \tilde{\boldsymbol{\omega}}_B \mathbf{v}_B$$

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# Rate of Change of Translational Position

Express derivative in inertial frame

$$\dot{\mathbf{r}}_I = \mathbf{v}_I = \mathbf{H}_B^I \mathbf{v}_B$$

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# Rate of Change of Angular Orientation (Euler Angles)

- **Body-axis angular rate vector components are orthogonal**

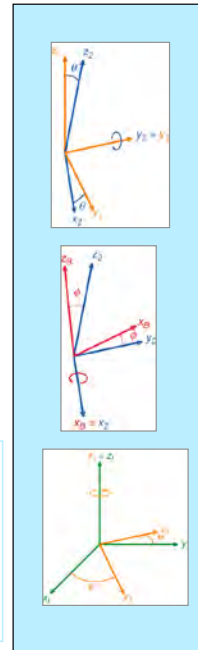
$$\omega_B = \begin{bmatrix} \omega_x \\ \omega_y \\ \omega_z \end{bmatrix}_B \triangleq \begin{bmatrix} p \\ q \\ r \end{bmatrix}$$

- **Euler angles form a non-orthogonal vector**

$$\Theta \triangleq \begin{bmatrix} \phi \\ \theta \\ \psi \end{bmatrix}$$

- **Therefore, Euler-angle rate vector is not orthogonal**

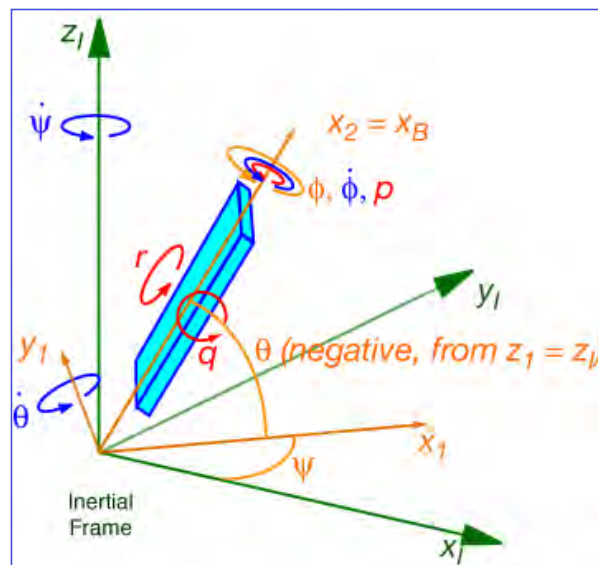
$$\frac{d\Theta}{dt} = \begin{bmatrix} \dot{\phi} \\ \dot{\theta} \\ \dot{\psi} \end{bmatrix} \neq \begin{bmatrix} \omega_x \\ \omega_y \\ \omega_z \end{bmatrix}_I \neq \omega_I$$



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## Transformation From Euler-Angle Rates to Body-Axis Rates

- $\dot{\psi}$  is measured in the Inertial Frame
- $\dot{\theta}$  is measured in Intermediate Frame #1
- $\dot{\phi}$  is measured in Intermediate Frame #2



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# Sequential Transformations from Euler-Angle Rates to Body-Axis Rates

$\dot{\psi}$  is measured in the Inertial Frame

$\dot{\theta}$  is measured in Intermediate Frame #1

$\dot{\phi}$  is measured in Intermediate Frame #2

$$\begin{bmatrix} p \\ q \\ r \end{bmatrix} = \mathbf{I}_3 \begin{bmatrix} \dot{\phi} \\ 0 \\ 0 \end{bmatrix} + \mathbf{H}_2^B \begin{bmatrix} 0 \\ \dot{\theta} \\ 0 \end{bmatrix} + \mathbf{H}_2^B \mathbf{H}_1^2 \begin{bmatrix} 0 \\ 0 \\ \dot{\psi} \end{bmatrix}$$

... which is

$$\begin{bmatrix} p \\ q \\ r \end{bmatrix} = \begin{bmatrix} 1 & 0 & -\sin\theta \\ 0 & \cos\phi & \sin\phi\cos\theta \\ 0 & -\sin\phi & \cos\phi\cos\theta \end{bmatrix} \begin{bmatrix} \dot{\phi} \\ \dot{\theta} \\ \dot{\psi} \end{bmatrix} \triangleq \mathbf{L}_I^B \dot{\Theta}$$

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# Inversion to Transform Body-Axis Rates to Euler-Angle Rates

Transformation is not orthonormal

$$\mathbf{L}_I^B = \begin{bmatrix} 1 & 0 & -\sin\theta \\ 0 & \cos\phi & \sin\phi\cos\theta \\ 0 & -\sin\phi & \cos\phi\cos\theta \end{bmatrix}$$

Inverse transformation is not the transpose

$$\left(\mathbf{L}_I^B\right)^{-1} \triangleq \mathbf{L}_B^I \neq \left(\mathbf{L}_I^B\right)^T$$

$$\left(\mathbf{L}_I^B\right)^{-1} = \begin{bmatrix} 1 & 0 & -\sin\theta \\ 0 & \cos\phi & \sin\phi\cos\theta \\ 0 & -\sin\phi & \cos\phi\cos\theta \end{bmatrix}^{-1} = \frac{Adj\left(\mathbf{L}_I^B\right)}{\det\left(\mathbf{L}_I^B\right)}$$

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## Euler-Angle Rates

$$(\mathbf{L}_I^B)^{-1} = \frac{Adj(\mathbf{L}_I^B)}{\det(\mathbf{L}_I^B)} = \begin{bmatrix} 1 & 0 & -\sin\theta \\ 0 & \cos\phi & \sin\phi\cos\theta \\ 0 & -\sin\phi & \cos\phi\cos\theta \end{bmatrix}^{-1} = \begin{bmatrix} 1 & \sin\phi\tan\theta & \cos\phi\tan\theta \\ 0 & \cos\phi & -\sin\phi \\ 0 & \sin\phi\sec\theta & \cos\phi\sec\theta \end{bmatrix}$$

Euler-angle rates from body-axis rates

$$\begin{bmatrix} \dot{\phi} \\ \dot{\theta} \\ \dot{\psi} \end{bmatrix} = \begin{bmatrix} 1 & \sin\phi\tan\theta & \cos\phi\tan\theta \\ 0 & \cos\phi & -\sin\phi \\ 0 & \sin\phi\sec\theta & \cos\phi\sec\theta \end{bmatrix} \begin{bmatrix} p \\ q \\ r \end{bmatrix} = \mathbf{L}_B^I \boldsymbol{\omega}_B$$

Can the inversion become singular?  
What does this mean?



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## Summary of Six-Degree-of-Freedom (Rigid Body) Equations of Motion

*Rigid-body dynamic equations are nonlinear*

$$\begin{aligned} \dot{\mathbf{r}}_I &= \mathbf{H}_B^I \mathbf{v}_B \\ \dot{\mathbf{v}}_B &= \frac{1}{m} \mathbf{f}_B - \tilde{\boldsymbol{\omega}}_B \mathbf{v}_B \\ \dot{\boldsymbol{\Theta}} &= \mathbf{L}_B^I \boldsymbol{\omega}_B \\ \dot{\boldsymbol{\omega}}_B &= \mathbb{I}_B^{-1} (\mathbf{m}_B - \tilde{\boldsymbol{\omega}}_B \mathbb{I}_B \boldsymbol{\omega}_B) \end{aligned}$$

**Translational position  
and velocity**

$$\mathbf{r}_I = \begin{bmatrix} x \\ y \\ z \end{bmatrix}; \quad \mathbf{v}_B = \begin{bmatrix} u \\ v \\ w \end{bmatrix}$$

**Rotational position  
and velocity**

$$\boldsymbol{\Theta} = \begin{bmatrix} \phi \\ \theta \\ \psi \end{bmatrix}; \quad \boldsymbol{\omega}_B = \begin{bmatrix} p \\ q \\ r \end{bmatrix}$$

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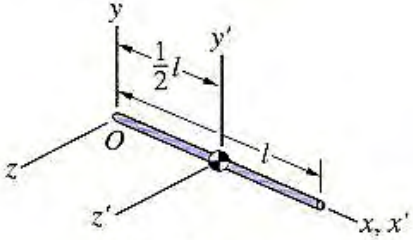
***Next Time:  
Flying and Swimming Robots***

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***Supplemental  
Material***

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# Moments and Products of Inertia for Common Constant-Density Objects are Tabulated

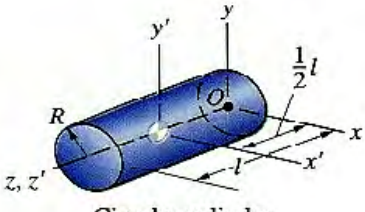


Slender bar

$$I_{x \text{ axis}} = 0, \quad I_{y \text{ axis}} = I_{z \text{ axis}} = \frac{1}{3} ml^2,$$

$$I_{xy} = I_{yz} = I_{zx} = 0.$$

$$I_{x' \text{ axis}} = 0, \quad I_{y' \text{ axis}} = I_{z' \text{ axis}} = \frac{1}{12} ml^2,$$

$$I_{x'y'} = I_{y'z'} = I_{z'x'} = 0.$$


Circular cylinder

$$\text{Volume} = \pi R^2 l$$

$$I_{x \text{ axis}} = I_{y \text{ axis}} = m \left( \frac{1}{3} l^2 + \frac{1}{4} R^2 \right), \quad I_{z \text{ axis}} = \frac{1}{2} m R^2,$$

$$I_{xy} = I_{yz} = I_{zx} = 0.$$

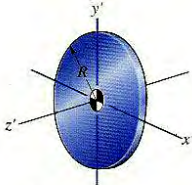
$$I_{x' \text{ axis}} = I_{y' \text{ axis}} = m \left( \frac{1}{12} l^2 + \frac{1}{4} R^2 \right), \quad I_{z' \text{ axis}} = \frac{1}{2} m R^2,$$

$$I_{x'y'} = I_{y'z'} = I_{z'x'} = 0.$$

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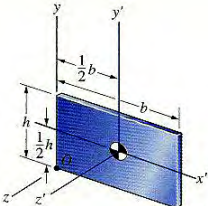
## Moments and Products of Inertia

(Bedford & Fowler)



Thin circular plate

$$I_{x' \text{ axis}} = I_{y' \text{ axis}} = \frac{1}{4} m R^2, \quad I_{z' \text{ axis}} = \frac{1}{2} m R^2,$$

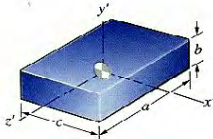
$$I_{x'y'} = I_{y'z'} = I_{z'x'} = 0.$$


Thin rectangular plate

$$I_{x \text{ axis}} = \frac{1}{3} m h^2, \quad I_{y \text{ axis}} = \frac{1}{3} m b^2, \quad I_{z \text{ axis}} = \frac{1}{3} m (b^2 + h^2),$$

$$I_{xy} = \frac{1}{4} m b h, \quad I_{yz} = I_{zx} = 0.$$

$$I_{x' \text{ axis}} = \frac{1}{12} m h^2, \quad I_{y' \text{ axis}} = \frac{1}{12} m b^2, \quad I_{z' \text{ axis}} = \frac{1}{12} m (b^2 + h^2),$$

$$I_{x'y'} = I_{y'z'} = I_{z'x'} = 0.$$


Rectangular prism

$$\text{Volume} = abc$$

$$I_{x' \text{ axis}} = \frac{1}{12} m (a^2 + b^2), \quad I_{y' \text{ axis}} = \frac{1}{12} m (a^2 + c^2),$$

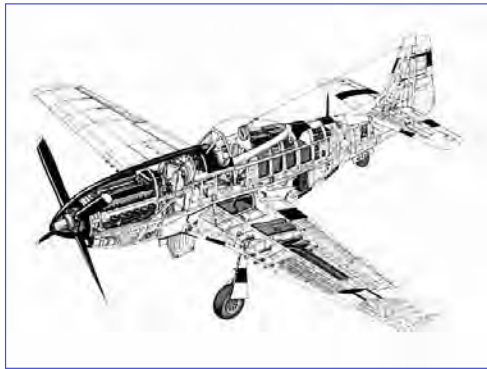
$$I_{z' \text{ axis}} = \frac{1}{12} m (b^2 + c^2), \quad I_{x'y'} = I_{y'z'} = I_{z'x'} = 0.$$

$$\mathbb{I} = \begin{bmatrix} I_{xx} & -I_{xy} & -I_{xz} \\ -I_{xy} & I_{yy} & -I_{yz} \\ -I_{xz} & -I_{yz} & I_{zz} \end{bmatrix}$$

## Construction of Inertia Matrix

Build up moments and products of inertia from components using **parallel-axis theorem**, e.g.,

$$I_{xx_{airplane}} = I_{xx_{wings}} + I_{xx_{fuselage}} + I_{xx_{horizontal\ tail}} + I_{xx_{vertical\ tail}} + \dots$$



... or use software, e.g.,  
**Autodesk, Creo**  
**Parametric, SimMechanics,**  
...