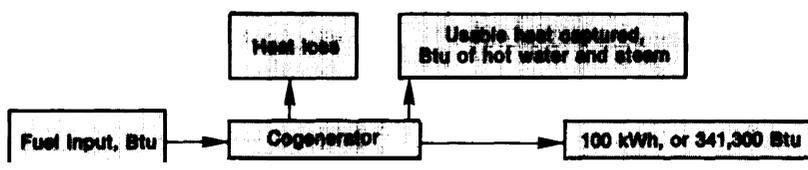


Appendix B Emissions Changes

A number of analyses of emissions changes caused by cogeneration have been conducted. All of these analyses, however, are site-specific and do not illustrate the effects of changing critical assumptions, or they have extrapolated to regional or national emissions changes by using simplifying assumptions about critical parameters (e.g., the type of fuels "backed out" of the utilities' centralized systems) that may be significantly in error.

The following tables display the emissions attributable to each component in a possible switch from a system using central station-generated electricity and a local heat source to a system using cogeneration. A number of options for the central plant, cogenerator, and heat source are shown in order to allow a range of circumstances to be evaluated. Because of the significant variation possible in each of the components (for example, the thermal efficiencies and emissions

Table B-I.—Emissions From Cogenerator Options



Type	Fuel Input	Heat captured ^a	Thermal efficiency	Emissions (lb/100 kWh)					Comment
				NO _x	Particulate	CO	HC	SO ₂ ^b	
Diesel	990,000 ^c	350,000 ^c	0.70 ^d	3.43 ^e	0.07 ^f	owe	0.10 ^g	0.29	Oil, Uric.
				2.49 ^g	ND	0.64 ^h	0.87 ^h	0.02 ^h	Dual Fuel, Uric.
				2.20 ^h	NS	NS	NS	NS	NSPS
Gas turbine	1,365,000	610,000	0.70 ^d	1.2 ^h	0.03 ^m	0.15 ^m	0.05 ^m	0.01 ^m	Gas, Unc.
				0.4 ^h	NS	NS	NS	1.09	Oil, Unc.
				2.08 ^h	0.30 ^h	0.12	0.03	3.56 ^h	NSPS
NSPS steam turbine.	2,970,000 ^p	2,000,000 ^p	0.79 ^p	0.89 ^h	0.30 ^h	NEG	0.03	2.38 ^q	Coal
				0.59 ^h	0.03	NEG	0.12	0.21	oil
									Gas

ND - No data found
NS - No standard
NEG - Negligible

^aUnless the cogenerating system has heat storage capability and/or very careful balancing of heat production and actual need, less heat than this will be usefully captured, the system efficiency will decrease, and the overall emissions balance between cogeneration and the central station power/local heating source will worsen. b values for SO₂ are entirely fuel dependent. Essentially 100 percent of the sulfur contained in the fuel is transformed into SO₂ upon combustion.

^cBased on fuel rate data in Environmental Protection Agency, Standards Support and Environmental Impact Statement for Stationary Internal Combustion Engines, draft, EPA-450/2-78-125a July 1979, assuming 95 percent generator efficiency.

^dThese data sources did not converge on any efficiency value as "best", but values ranged from 62 to 80 percent. The major source of variability appears to be the amount of heat captured rather than the total fuel input.

^eBased on sales-weighted averages for large-bore diesels, data in EPA, op. cit. (note c).

^fEnvironmental Protection Agency, *Compilation of Air Pollutant Emission Factors (AP-42)*, 1978.

^gAssumes 0.2 percent sulfur diesel fuel or distillate oil.

^hThe New Source Performance Standard for diesels burning oil or oil/natural gas combinations is 600 ppm of NO_x. This is roughly translatable into about 7 grams per horsepower hour, or about 2.2 lb/MMBtu, personal communication from Douglas Bell, Office of Air Quality Planning and Standards, Research Triangle Park, N.C. The application of NO_x emission controls may have an effect on emissions of other pollutants. Because efficiency may decrease somewhat with such controls the effect on CO and HC may be adverse.

ⁱTotal fuel input and heat captured in a gas turbine cogenerator are extremely variable. Data shown are from ICF, Inc., *A Technical and Economic Evaluation of Dispersed Electric Generation Technologies*, draft final report to OTA, October 1980, table 3-3, simple-cycle turbine.

^kibid., pages. With the "typical" turbine in General Accounting Office, *Industrial Cogeneration—What It Is, How It Works, Its Potential*, EMD-80-7, Apr. 29, 1960.

^lEnvironmental Protection Agency, *Standards Support and Environmental Impact Statement for Stationary Gas Turbines*, EPA-450/2-77-017a, September 1977, pp. 3-110, for "typical," uncontrolled turbines.

^mEnvironmental protection Agency, AP-42, op. cit. (note f). Note that the AP-42 value for NO_x is 0.6 lb/MMBtu v. 0.6 lb/MMBtu for EPA, OP. cit. (note l).

ⁿFew data were found. This value applies to a GE 7821 B combustion turbine, cited in J. A. Taylor, *An Air @ @- Assessment for ICES Options*, Argonne National Laboratory, September 1960, draft.

^oThe New Source Performance Standard for gas turbines is 75 ppm of NO_x, roughly translatable into about 0.225 to 0.3 lb/MMBtu, personal communication from Douglas Bell, OAQPS, RTP, N.C. Table 3-11 in EPA, op. cit. (note 1) equates 75 ppm at 15 percent oxygen to 0.3 lb/MMBtu, but the significant variability in fuel rates of gas turbines implies a range of "lb/MMBtu" rates.

^pFrom General Accounting Office, 1980, op. cit., (note k), p. 92. Because a steam turbine may be designed to convert anywhere from zero to over 30 percent of its fuel energy to electricity, these values represent only one possible combination in an extremely broad range.

^q40 CFR 60 subpt. D, NSPS for steam generators other than utility over 73 MW input. Generators smaller than this size are subject to State implementation plan regulations.

SOURCE: Office of Technology Assessment.

from a gas turbine or diesel can vary over a fairly wide range), however, the tables capture only a portion of the potential variability in emissions balances.

The values of energy flow and emissions displayed are normalized to an "electrical output of 100 kilowatt-hours. Emission "balances" for particular combina-

tions of cogenerator, central power facility, and local heat source can be calculated by using the formula:

$$\begin{aligned} & \text{(net emissions in lbs/100 kWh} \\ & \text{of cogenerated power)} = \text{(cogenerator emissions, table B-1)} \\ & \quad - \text{(central station power emis-} \\ & \quad \text{sions, table B-2) - (hot water and} \\ & \quad \text{steam emission factor, table B-3)*} \\ & \quad \text{(heat captured, table B-1/1 O.)} \end{aligned}$$

Table B-2.—Emissions From Central Station Power Stations (to provide 100 kWh of delivered power)

Type	Fuel input	Emissions ^a , lb/100 kWh				
		NO _x	Part	CO	HC	SO _x
Coal-fired powerplant, NSPS, scrubber	1,100,000 ^b	0.55	0.03	0.04	0.01	1.32 ^c
Older coal-fired plant, 3% sulfur IO% ash with 95% control, 13,000 Btu/lb	1,000,000	0.69	0.31	0.04	0.01	4.38
Oil-fired plant, NSPS, low sulfur oil.	1,000,000	0.30	0.03	0.04	0.01	0.80
Older oil-fired plant, 1 % sulfur.	1,000,000	0.70	0.05	0.04	0.01	1.05
Older natural gas-fired plant	1,000,000	0.67	0.01	0.02	0.04	NEG.
Existing gas turbine peaking plants	1,100,000 ^d	0.43	(.66) ^e 0.01	0.12	NEG.	NEG. Gas
Existing gas turbine peaking plants	1,100,000 ^d	0.53	(.99) ^e 0.04	0.12	0.04	0.03 Oil
NSPS gas turbine		0.3				

^aEmissions from the following source: 1) Compilation of Air Pollutant Emission Factors, Third Edition, Environmental Protection Agency, August 1977; and 2) Federal Regulations 40 CFR, Part 60, defining New Source Performance Standards for Fossil-fueled steam-electric powerplants.

^bThe higher heat rate is accounted for by the efficiency loss caused by the scrubber.

^cAssumes high sulfur coal. Requirement for continuous control systems achieving 70 to 90 percent efficiency would reduce SO_x emissions to as low as 0.6 lb/MMBtu for low to medium sulfur coals.

^dAlthough gas turbine rates are quite variable, the larger GE and Westinghouse turbines (over 50MW) tend to have fuel rates between 10,500 and 12,000 Btu/kWh, Environmental Protection Agency, Standards Support and Environmental Impacts Statement for Stationary Gas Turbines, EPA-450/77-017a, September 1977, pp. 3-46.

^eThe first values are those given in footnote a), above, the second are "typical" values for a range of turbines given in EPA, 1977, op. cit. (footnote d). An examination of turbine data (ibid., pp. 3-46) indicates that the larger utility turbines do not appear to emit nitrogen oxides at a lower unit rate than smaller industrial turbines. The larger emissions value is used to construct the emission balances.

SOURCE: Office of Technology Assessment.

Table B-3.—Emissions From Hot Water and Steam Systems (to provide 1,000,000 Btu of usable heat energy)

Heat source	Fuel input	NO _x	Emissions, lb/10 ⁶ Btu usable heat				
			Particulate	CO	HC	SO _x	
Furnace and hot water heater	1,250,000 Btu	0.12	0.01	0.02	0.01	NEG.	Gas
		0.16	0.02	0.04	0.01	0.24	Oil (.2%S)
NSPS steam boilers.	1,250,000 Btu	0.37	0.13	0.05	0.01	1.50	Coal
		0.25	0.13	0.04	0.01	1.00	Oil
		0.25	0.01	0.02	NEG.	NEG.	Gas
Small (<250 x 10 ⁶ Btu/hr) industrial boiler	1,250,000 Btu	0.72	0.63	0.10	0.05	3.65	Coal ^a
		0.50	0.19	0.04	0.01	1.31	Oil (1%S)
		0.21	0.02	0.02	NEG.	NEG.	Gas

^a 10 percent ash, 2 percent sulfur, 13,000 Btu/lb, 90 percent particulate control.

SOURCE: Office of Technology Assessment.