Particle Simulation of Vibrated Gas-Fluidized Beds of Cohesive Fine Powders

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We have used three-dimensional particle dynamics simulations, coupled with volume-averaged gas phase hydrodynamics, to study vertically vibrated gas-fluidized beds of fine, cohesive powders. The volume-averaged interstitial gas flow is restricted to be one-dimensional (1D). This simplified model captures the spontaneous development of 1D traveling waves, which corresponds to bubble formation in real fluidized beds. We have used this model to probe the manner in which vibration and gas flow combine to influence the dynamics of cohesive particles. We have found that, as the gas flow rate increases, cyclic pressure pulsation produced by vibration becomes more and more significant than direct impact, and in a fully fluidized bed, this pulsation is virtually the only relevant mechanism. We demonstrate that vibration assists fluidization by creating large tensile stresses during transient periods, which helps break up the cohesive assembly into agglomerates.

1. Introduction

It is well-known that fine, cohesive particles cannot be fluidized easily.1,2 When fluidization by an upflow of gas is attempted, assembles of such particles have a tendency to lift up as a plug and only form ratholes and cracks through which the fluid escapes. Attractive interparticle forces frequently arise, as a result of capillary liquid bridges or van der Waals forces. A variety of techniques to achieve smooth fluidization of such particles has been explored in the literature. When the cohesion arises from van der Waals forces, as in the case of Geldart type C particles, which are typically 30 μm or smaller in size,4 coating with hard nanoparticles5 or an ultrathin film6 can weaken the attraction between the bed particles, thus enabling smooth fluidization. Alternate approaches to facilitate fluidization include causing agglomerate breakup through a secondary supply of energy using mechanical vibration,5,12 acoustic waves,13-15 or an oscillating magnetic field.16 Such approaches have been shown to be effective, even for beds of nanoparticles.11 In the present study, we are concerned with some aspects of the mechanics of vibrated fluidized beds of cohesive particles.

Vibrated layers of granular materials have been studied extensively in the literature, and the formation of patterns in shallow granular layers is now well-known.17,18 Mixing and segregation in such vibrated beds have also received considerable attention.19-21 The motion of large intruders in vibrated beds, leading to the well-known Brazil nut and reverse Brazil nut effects,20,22 has also received much attention in the literature, where experimental measurements suggest a non-negligible influence of the interstitial gas on the observed flow patterns.23,24 Generally, the deeper the bed and/or the smaller the bed particles, the greater the influence of the interstitial gas phase on the dynamics of the assembly of particles.

In vibrated fluidized beds, where vibration is supplemented with a fluidizing gas flow (or vice versa), the importance of the interstitial gas is obvious, because the drag that is due to the gas flow supports a substantial portion of the weight of the particles. Vibrated fluidized beds have found many applications in industrial practice (e.g., see an article by Squires25). Understanding the manner in which the vibration aids the fluidization process is important both for macroscopic analysis of vibrated fluidized beds and for detailed interrogation of agglomerate size distribution and mixing at the agglomerate and particle scales.

Predicting the minimum fluidization velocity of a vibrated fluidized bed (of cohesive particles) is perhaps the simplest quantitative, macroscopic analysis problem one can think of. This indeed has been the subject of many investigations,4,5,7-9,11,26-29 and different approaches have been proposed in the literature to incorporate the effect of vibration on the overall force balance used to determine the minimum fluidization velocity. Musters and Rietema26 suggested that additional terms must be included in the force balance relation, to account for the increased pressure drop due to cohesion. They included additional terms for interparticle forces, as well as wall friction. Liss et al.27 proposed an additional term to account for the effect of cohesion arising from liquid bridges. Wank et al.8 showed that the agglomerate size decreases as the vibration intensity increases, and they studied the effect of the pressure on the minimum fluidization velocity. Erdész and Mujumdar28 developed a theory that includes the effect of vibration in the prediction of the minimum fluidization velocity; they found that the pressure drop decreased as the vibration intensity in their experiments increased with various particles in the size range of 0.15–2.75 mm. However, Tarisin and Anuar29 found the opposite trend in their study of vibrofluidization of particles of 1–34 μm, and others7,9,11 found no appreciable dependence on the vibration intensity with particles ranging in size from 12 nm to 100 μm.

Such conflicting reports stem from our limited understanding of the manner in which the vibrating boundary interacts with the assembly of particles in the bed. One can readily envision at least two modes of interaction: (a) when the effect of the interstitial gas on particle motion is of negligible importance, the vibrating base plate clearly imparts impulse to the particle assembly periodically only through direct collisions; and (b) when gas-particle drag is non-negligible, a periodically varying pressure field can be expected to develop in the bed, and the base plate interacts with the bed particles indirectly by driving these pressure pulsations as well. The relative importance of these two contributions can be expected to be dependent on the bed depth, the particle size, and the superficial velocity of the fluidizing gas; however, quantitative estimates are unavailable.

At a more detailed level, it is readily apparent that vibration results in the formation of small agglomerates, which are more

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amenable to fluidization; yet, how such small agglomerates are formed is not understood. A clear understanding of this mechanism is an important first step in any effort to capture the effect of vibration on the dynamics of agglomerates (e.g., in a population-balance-type model for vibrated fluidized beds). At first glance, one may speculate that vibration drives vigorous collisions between agglomerates, which, in turn, causes their breakup; however, it is not known if this is, indeed, the dominant mechanism.

In the present study, we have examined the interaction of the vibrating base plate with a bed of particles through detailed simulations. We will demonstrate that (i) as the fluidizing gas velocity is increased, the interaction with the boundary through pressure pulsation becomes more and more dominant, and (ii) above the minimum fluidization conditions, the interaction occurs almost exclusively through pressure pulsations. We will also show that the large tensile stress induced by the vibration is a more important mechanism (than vigorous collisions between agglomerates) in causing the breakup of the cohesive particle assembly.

The dynamic behavior of gas-fluidized beds containing a large number of particles has been widely examined in the literature through two-fluid models. Considerable progress has been made in developing and validating the closures for assemblies of noncohesive particles, but not for the case of cohesive materials. Accurate two-fluid model boundary conditions that capture the effect of vibrating boundaries are also unavailable. Furthermore, the two-fluid model approach is not well-suited for investigation of the mechanism of agglomerate breakup. For these reasons, we model vertically vibrated gas-fluidized beds of fine powders using a hybrid scheme, where the solid phase is treated as discrete spherical particles, following the so-called discrete element method (DEM, or soft-sphere molecular dynamics), whereas the gas phase is treated as a continuum, following the volume-averaged hydrodynamics, just as in a two-fluid model. This DEM-based hybrid approach was proposed by Tsuji et al.44 in their simulation of two-dimensional (2D) fluidized beds, and it has been subsequently refined by others. Recently, this model has been used to study fluidized beds of Geldart A particles.45 The effect of an arbitrary cohesive force that is proportional to the particle buoyant weight, and segregation in fluidized beds of bidisperse particles.

Even with the high-speed computing that is available today, one can only simulate small systems, which are orders of magnitude smaller than real vibrated fluidized beds, through this hybrid approach. Therefore, it is essential that one choose suitable, idealized problems to probe the underlying mechanics. With this in mind, we have chosen essentially one-dimensional (1D) vibrated fluidized beds of particles, applying periodic boundary conditions in the two lateral directions. This limits the macroscopic dynamics to the vertical direction only, and one obtains one-dimensional traveling waves (1D-TW) instead of bubblelike voids observed in experiments and in fully three-dimensional flow simulations; nevertheless, such an idealized problem is, in our opinion, adequate to expose the manner in which the vibrating base plate interacts with the bed particles and how dense cohesive regions are broken down into smaller agglomerates.

The rest of the paper is organized as follows. The DEM-based hybrid model is described in section 2, and bubble formation, realized as 1D-TW in our geometry (narrow cross-sectional areas), is presented in section 3.1. The effect of cohesion and vibration on the fluidization is presented in section 3.2, and the manner in which the vibration enhances the fluidization will be discussed in section 3.3. The pressure drop in vibrated fluidized beds and the mechanism through which the vibration breaks up cohesive assemblies will be presented in sections 3.4 and 3.5, respectively, which are followed by the conclusions in section 4.

2. Method: Discrete Element Method (DEM)-Based Hybrid Model

Since its introduction by Cundall and Strack almost three decades ago, the DEM has been successfully used in modeling various particulate flow problems, including hopper flows, shearing cells, rotating drums, and oscillated layers. Comprehensive description of this method can be found in the literature. Below, we briefly describe the main idea of the DEM, and subsequently focus on how the volume-averaged gas-phase hydrodynamics is coupled with the individual particle dynamics in our model.

2.1. Discrete Element Method (DEM). In the DEM simulation, particles are modeled as “soft” spheres (the deformation taken into account by overlaps), whose trajectories are computed by integrating Newton’s equations of motion. When objects (particles or system boundaries) come into contact, the interaction is resolved by decomposing the interaction force into the normal and tangential directions, relative to the displacement vector between the objects at contact ($F_{\text{cont}} = (F_{\text{n}}, F_{\text{t}})$), and the energy dissipation upon contact is characterized by the inelasticity and the surface friction. We use the so-called spring-dashpot model, with a Hookean spring. The objects are allowed to overlap upon contact, and the contact force in the normal direction ($F_{\text{n}}$) is determined by the amount of overlap ($\Delta_n$) and the normal component of the relative velocity at contact $v_n$:

$$F_{\text{n}} = (k_n \Delta_n - \gamma_n v_n) h$$

where $k_n$ is the spring stiffness in the normal direction, $\gamma_n$ the damping coefficient, and $h$ the unit vector in the normal direction at contact, pointing from the contact point toward the particle center. The damping coefficient $\gamma_n$ is related to $k_n$ by the normal coefficient of restitution $e$ ($0 \leq e \leq 1$):

$$\frac{4k_n/m^*}{(\gamma_n/m^*)^2} = 1 + \left(\frac{\pi}{\log e}\right)^2$$

where $1/m^* = 1/m_i + 1/m_j$, and $i$ and $j$ are indices of interacting particles or objects. In principle, the value of $k_n$ is determined by the Young’s modulus of the material under consideration. However, unless stated otherwise, we use a much smaller value for $k_n$, compared to that computed based on the usual range of Young’s modulus. If the main results are not qualitatively different, it is favorable to use a smaller value of $k_n$, because the collision duration time in DEM scales with $k_n^{-1/2}$, which determines the integration time-step size required to accurately resolve the interaction during the contact. This is a well-known issue in DEM simulations. We varied $k_n$ over 3 orders of magnitude and verified that the main results are not dependent sensitively on the choice of $k_n$, even though the actual contact force between the objects certainly is dependent on the value of $k_n$. For instance, the results for two $k_n$ values differing by a factor of 10 will be presented in Figures 7–9 later in this paper.
The interaction in the tangential direction is modeled by a "spring and slider", and the contact force is given by

\[ F_s = -\text{sign}(\nu_s) \times \min(k_s,\mu_s|F_s|)\hat{s} \]  

(3)

where \( \nu_s \) is the tangential component of the relative velocity at contact \( (\nu_s = \nu \hat{t} \times (v \times \hat{t})) \) and \( \hat{s} \) is the unit vector in the tangent plane collinear with the component of the relative velocity at contact). \( k_s \) is the tangential spring stiffness that is related to \( k_s \) by the Poisson’s ratio of the material \( v_p \) (where \( k_s = 2k_g(1 - \nu_p)(2 - \nu_p) \)), and \( \Delta s \) is the magnitude of tangential displacement from the initial contact. The magnitude of the total tangential force is limited by the Coulomb frictional force \( \mu_s|F_s|\), where \( \mu_s \) is the coefficient of friction. More sophisticated and/ or realistic interaction models, such as that of Walton and Braun’s model49 or a Hertzian spring-dashpot model,50 may also be used. However, we choose a simple Hookean spring-dashpot and spring-slider model, because it has been shown to successfully reproduce many experimental observations.41,43,45 and it is computationally more tractable than others.

Among different interparticle forces, other than those due to contact, we consider only cohesion that results from van der Waals forces. In principle, the cohesion can be dependent on the particle characteristics, such as polarizability, particle size, and asperity.51 However, we adopt a simple formula by Hamaker, because we intend to determine the effect of cohesion on the fluidization behavior, rather than to validate different cohesion models. Particulate flows in industry often consist of particles with a wide range of sizes and shapes; however, we seek a better understanding of simple systems that consist of monodisperse spheres, which are well-characterized by a small set of parameters. The cohesive van der Waals force between two spheres of radii \( r_i \) and \( r_j \) can be expressed as\(^{51}\)

\[ F_c = -\frac{A \cdot 2r_s (s + r_i + r_j)}{3 \cdot [s + 2r_s + 2r_i]} \times \left[ \frac{s^2 (s + 2r_i + 2r_j)}{(s + r_i + r_j)^2} \right] \left( 1 - \frac{r_j}{r_i} \right)^2 \hat{n} \]

\[ \approx -\frac{A \cdot r_s}{12 \cdot s^2} \hat{n} \quad \text{(for } r_i = r_j \text{ and } s \ll r) \]

(4)

where \( A \) is the Hamaker constant and \( s \) is the intersurface distance. Because the original formula is a rapidly decreasing function of \( s \), further simplification using the assumption of \( s \ll r \) has been made. This model has a singularity at contact. To avoid this artifact, we introduce a widely accepted minimum cutoff value for the intersurface distance of 0.4 nm (\( \equiv \delta^* \)), which corresponds to the intermolecular center-to-center distance.\(^{52}\) In what follows, the level of cohesion is represented by the cohesive Bond number \( Bo \), which is defined as the ratio of the maximum cohesive force (at the minimum cutoff separation \( \delta^* \)) to the particle weight. Other types of cohesion can be readily taken into account in DEM-based models.\(^{39,53,54}\)

### 2.2. Coupling with Gas-Phase Hydrodynamics.

The dynamics of individual particles is coupled with the volume-averaged gas-phase hydrodynamics. In this hybrid model, the equations of motion for individual particles have two additional terms (compared to traditional DEM modeling particles under vacuum) arising from the presence of the gas-phase:

\[ m_p \frac{dv_p}{dt} = m_p g_{\text{eff}} + F_{\text{cont}} + F_c + \frac{V_p}{\phi} \beta(\phi)(u_g - v_p) - V_p \nabla p \]

(5)

where \( m_p \) and \( v_p \) are individual particle mass and velocity, respectively. The first term on the right-hand side represents the body force due to gravity, and \( g_{\text{eff}} \) is the effective gravitational acceleration in the reference frame where equations are integrated. For nonvibrated beds, \( g_{\text{eff}} \) is simply the gravitational acceleration \( g \). When the bed is subject to a single frequency oscillation, the equations are integrated in the vibrated frame, and \( g_{\text{eff}} = g[1 + \Gamma \sin(\omega t)] \), where \( \Gamma \) is the maximum acceleration of the base plate (distributor) nondimensionalized by the gravitational acceleration (\( \Gamma = A_p(2\pi f)^2/\omega^2 g \), where \( g \) is the gravitational acceleration, \( \omega \) is the oscillation amplitude, and \( f \) is the oscillation frequency). We assume the oscillating base plate is comprised of the same materials as the particles (the same values for \( e \) and \( p \)), and that the mass of the plate is infinitely large, compared to that of an individual particle. The second term and the third term represent the aforementioned contact force and van der Waals force, respectively. The total force acting on the particles due to the fluid is commonly partitioned into the local drag component and the effective buoyant component, as done here (see, e.g., an article by Ye et al.37). The fourth term accounts for the drag force, and the last term accounts for the contribution of the gradually varying part of the pressure field, where \( V_p \) is the volume of each particle; \( \phi \) and \( u_g \) are volume-averaged solid-phase volume fraction and gas-phase velocity, respectively; \( \beta \) is the inter-phase momentum transfer coefficient;\(^{31}\) and \( p \) is the gas-phase pressure.

Generally, the gas-phase quantities are obtained by simultaneously integrating the coarse-grained mass and momentum balance equations. We assume the gas phase to be incompressible, which will be validated later (see section 3.4). The addition of continuity equations for the gas phase and solid phase reads

\[ \nabla \cdot [(1 - \phi)u_g + \phi u_s] = 0 \]

(6)

and a reduced momentum balance equation for the gas phase, based on generalized Darcy’s law, is given as

\[ 0 = -(1 - \phi) \nabla p + \beta(\phi)(u_s - u_g) \]

(7)

where \( u_s \) is the coarse-grained solid-phase velocity. Coarse-grained variables are considered only on grids where the continuum balance equations are solved. Note that the solid phase continuum (or coarse-grained) variables are explicitly available in the course of DEM computation.

### 2.3. Beds of Narrow Cross-Sectional Area.

Only beds of narrow cross-sectional areas will be considered, and the volume-averaged gas phase (and, hence, the solid phase coarse-grained variables as well) is assumed to be 1D. However, the solid phase is maintained to be 3D, as the way particles pack and collide in lower dimensions are considerably different from those in realistic 3D cases. Our assumption allows us to consider relatively deep beds (through inexpensive computational effort) and to derive the basic physics of more-complicated dynamics in higher dimensions.

Solid-phase coarse-grained variables at 1D discrete grid points are computed by distributing the particle mass and momenta to the nearest two grid points, using a halo function \( h \) that continuously decreases to zero around the particle:

\[ h(z - z_0) = \begin{cases} 
\frac{1 - |z - z_0|}{\Delta z} & \text{for } |z - z_0| < \Delta z \\
0 & \text{otherwise}
\end{cases} \]

(8)

where \( z \) is the particle position in the vertical direction, \( z_0 \) is the position of a neighboring grid point, and \( \Delta z \) is the grid...
spacing. It is readily seen that \( h \) has the property that the particle quantities are distributed to the two nearby grid points, inversely proportional to the distance to the grid point. The coarse-grained variables, the number density \( n \) and \( u_s \), on the grids are then defined simply as

\[
n(z_0) = \sum_{i=1}^{N} h(z_i - z_0) \tag{9}
\]

\[
n(z_0) u_s(z_0) = \sum_{i=1}^{N} h(z_i - z_0) v_{p,i} \tag{10}
\]

where \( z_i \) and \( z_0 \) are the \( i \)th particle location and nearby grid location, respectively.

The assumption of the gas phase to be 1D facilitates further mathematical simplifications of the above particle–gas interaction formulation. In 1D continuum cases, eq 6 can be integrated:

\[
(1 - \phi) u_g + \phi u_s = U_i \tag{11}
\]

where \( U_i \) is the superficial gas flow velocity. Strictly speaking, in a vibrated fluidized bed, \( U_i \) may also vary periodically. The extent of its variation will be dependent on the dynamics of the gas in the plenum and the flow resistance offered by distributor (base plate). One can show that the temporal variation of \( U_i \) will be small for a highly resistive distributor plate, which we assume. Thus, in the modeling of both nonvibrated and vibrated fluidized beds, \( U_i \) will be considered to be a time-independent parameter.

After some manipulation, eq 5 can be rewritten as follows:

\[
m_p \frac{d v}{dt} = m_p g_{eff} + F_{cont} + F_c + \frac{V_p}{\phi} \beta(\phi) \times \left[ (u_s - v_p) - \frac{1}{(1 - \phi)^2} (u_s - U)_i \right] \tag{12}
\]

Note that the presence of the gas phase is realized as additional terms involving coarse-grained variables, instead of separate continuum equations to be integrated simultaneously. In the course of integration, \( \phi \) and \( u_s \) in eq 12 must be evaluated at the particle location, rather than at the grid points. We evaluate them by linearly interpolating those values at the neighboring grid points.

For the interphase momentum transfer coefficient \( \beta \), we use an expression proposed by Wen and Yu: 55

\[
\beta = \frac{3}{4} C_D \frac{\rho_g (1 - \phi) |u_g - u_s|}{d_p} (1 - \phi)^{2.65} \tag{13}
\]

where \( C_D \) is the drag coefficient, \( \rho_g \) is the gas-phase mass density, and \( d_p \) is the particle diameter. The drag coefficient proposed by Rowe66 is used in our model:

\[
C_D = \begin{cases} 
\frac{24}{Re_g} (1 + 0.15 Re_g^{0.67}) & \text{(for } Re_g < 1000) \\
0.44 & \text{(for } Re_g \geq 1000) 
\end{cases} \tag{14}
\]

where

\[
Re_g = \frac{(1 - \phi) \rho_g d_p |u_g - u_s|}{\mu_g} \tag{15}
\]

and \( \mu_g \) is the gas-phase viscosity. Because we consider fine powders, \( Re_g \) is generally small, and we use the assumption \( Re_g \ll 1 \), which further simplifies \( \beta \) to

\[
\beta(\phi) = 18 \frac{\rho_g}{d_p^2} (1 - \phi)^{2.65} \tag{16}
\]

Casting eqs 12 and 16 in a dimensionless form, using \( \rho_s, d_p, \sqrt{g d_p}, \sqrt{d_p/g} \) as the characteristic density, length, velocity, and time, respectively, one obtains the following nondimensional groups (arrows indicate changes in the notation from dimensional to nondimensional variables that will be used henceforth):

for spring stiffness: \( k_n \leftarrow \frac{k_n}{\rho_s g d_p^2} \)

for superficial gas flow rate: \( U_s \leftarrow \frac{U_s}{\sqrt{g d_p}} \)

for scaled minimum separation distance: \( \delta \leftarrow \frac{\delta^*}{d_p} \)

for the cohesive bond number: \( Bo \equiv \frac{A}{4 \pi \rho_s g d_p^2 \Lambda^2} \)

for the Stokes number: \( St \equiv \frac{\rho_s g^{1/2} d_p^{1/2}}{\mu_g} \)

together with nondimensional parameters, namely \( \Gamma, f \equiv f \sqrt{d_p/g} \), \( e, \mu \), and \( v_p \). (See the Nomenclature section and Table 1 for a listing of the parameters used in this study.)

### 3. Results and Discussion

We simulate both nonvibrated and vibrated gas-fluidized beds of noncohesive or cohesive particles (0 ≤ Bo ≤ 50). We consider beds of narrow, square-shaped cross-sectional areas of \( 5d_p \times 5d_p \) or \( 10d_p \times 10d_p \) with a static depth of \( H_0 \approx 100d_p \) and \( H_0 \approx 200d_p \) (which consist of 3000 and 6000 particles, respectively, in beds with a cross section of \( 5d_p \times 5d_p \)). Periodic boundary conditions are imposed in both lateral directions, to avoid strong side-wall effects in beds with such a small aspect ratio. We check that our results are not sensitively dependent on a particular choice of cross-sectional area or the depth of the bed (e.g., see Figure 2). We mostly use a bed with dimensions of \( \sim 5d_p \times 5d_p \times 100d_p \) in the following computations, unless otherwise stated. We used values of \( d_p, 1.5d_p, \) and

<table>
<thead>
<tr>
<th>Parameter</th>
<th>Typical Values for Dimensional Quantities</th>
<th>Nominal Values for Dimensionless Parameters</th>
</tr>
</thead>
<tbody>
<tr>
<td>( \mu_g )</td>
<td>1.8 × 10^{-3} g/(m s)</td>
<td></td>
</tr>
<tr>
<td>( g )</td>
<td>981 cm/s²</td>
<td></td>
</tr>
<tr>
<td>( d_p )</td>
<td>50 μm</td>
<td></td>
</tr>
<tr>
<td>( \rho_g )</td>
<td>0.90 g/cm³</td>
<td></td>
</tr>
<tr>
<td>( \sqrt{g d_p} )</td>
<td>2.2 cm/s</td>
<td></td>
</tr>
<tr>
<td>( \sqrt{d_p/g} )</td>
<td>2.3 × 10^{-3} s</td>
<td></td>
</tr>
<tr>
<td>( \delta )</td>
<td>0.4 nm (= ( \delta^* ))</td>
<td>8.0 × 10^{-6}</td>
</tr>
<tr>
<td>( \Delta t )</td>
<td>5.6 × 10^{-7} s</td>
<td>2.5 × 10^{-4}</td>
</tr>
<tr>
<td>( \Delta z/d_p )</td>
<td>1.5</td>
<td></td>
</tr>
<tr>
<td>( \epsilon )</td>
<td>0.9</td>
<td></td>
</tr>
<tr>
<td>( \mu )</td>
<td>0.1</td>
<td></td>
</tr>
<tr>
<td>( k_n )</td>
<td>2.0 × 10^{-5}–2.0 × 10^{-6}</td>
<td></td>
</tr>
<tr>
<td>( v_p )</td>
<td>0.3</td>
<td></td>
</tr>
<tr>
<td>( \Gamma )</td>
<td>0–10</td>
<td></td>
</tr>
<tr>
<td>( f )</td>
<td>less than ~100 Hz</td>
<td>0–0.25</td>
</tr>
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</table>
2d₀ for the grid spacing Δz. The detailed profiles of the coarse-grained variables are slightly dependent on the choice of Δz (the bigger the grid size is, the smoother the variables are, as one can readily expect); however, the main results remain the same, unless the grid size is too large; we set Δz = 1.5d₀ in all the results presented here. In the following, all the quantities will be shown in nondimensional form.

3.1. Bubbling and One-Dimensional Traveling Waves. We start by considering conventional (nonvibrated) gas-fluidized beds of noncohesive particles to ensure that our model captures basic experimental observations. We first estimate the minimum fluidization velocity through a simulated quasi-static increase in the gas flow rate (from zero) and the measurement of the pressure drop across the bed, which yields \( \mathbf{U}_{\text{mf}} \approx 0.022 \). This estimate is slightly smaller than what we can compute using the force balance relation and the approximate formula of Wen and Yu (\( \mathbf{U}_{\text{mf}} \approx 0.023 \));\(^3^7\) in this calculation, we used a value of \( \phi_{\text{mf}} = 0.652 \), which was measured from the bulk of the bed. When the gas flow rate exceeds \( \mathbf{U}_{\text{mf}} \), the bed in our model starts to expand inhomogeneously and forms alternating bands of plugs and voids. Particles located at the bottom of one plug are “rain down” through a void region and accumulate at the top of the lower plug, causing the void regions to rise to the top in a periodic fashion (Figure 1). This phenomenon corresponds to the formation of a periodic train of bubblelike voids in real fluidized beds, which appears as 1D-TW in the narrow beds we consider. These waves represent the first stage in the bifurcation hierarchy, leading to various inhomogeneous structures in higher-dimensional (i.e., 2D or 3D) fluidized beds.\(^s^{58-60}\) The secondary bifurcations that occur in real fluidized beds are suppressed in our simulations, which are forced to retain the 1D character. As the gas flow rate increases, both the wave speed and amplitude increase (see Figure 1). In the subsequent sections, we will use the 1D-TW as an indicator that characterizes the fluidizability of the bed.

3.2. Cohesion, Vibration, and Fluidization. When the gas flow rate is well above \( \mathbf{U}_{\text{mf}} \), the bed exhibits clearly visible 1D-TW (for instance, see the cases of \( \mathbf{U}_s = 0.082 \) and 0.12 in Figure 1). In this section, we examine the influence of the cohesion (between particles) on the fluidizability of a bed, and we explore how mechanical vibration facilitates the fluidization of beds of cohesive particles.

Figure 2 shows the effect of \( \mathbf{Bo} \) on the wave speed, where we have kept \( \mathbf{St} \approx 55 \) and \( \mathbf{U}_s = 0.12 \) (the same values as in the last panel of Figure 1). As \( \mathbf{Bo} \) increases, the wave gradually slows and eventually disappears. At \( \mathbf{Bo} = 6 \), the wave travels intermittently, remaining stationary for some time and traveling at other times, and the wave speed during the nonstationary phase is shown in Figure 2. For \( \mathbf{Bo} > 7 \), the entire bed rises up as a plug at this flow rate, which is consistent with the well-known experimental observations in narrow beds of strongly cohesive particles (see the review article by Sundaresan\(^\text{60}\) and references therein). Further increases of \( \mathbf{U}_s \) improve the fluidizability of the bed only slightly, confirming that beds of highly cohesive particles cannot be fluidized by simply increasing the gas velocity.

The results shown in Figure 2 are obtained with the assumption that the interaction between the particle and the base plate is noncohesive. We check that the adhesion at the base plate does not make any difference in the above results, as well as all the main results presented here. The only noticeable difference is that strongly cohesive particles at the bottom of a bed become stuck to the plate for some time (during a cycle, in vibrated beds), depending on the oscillation parameters and the superficial gas flow velocity. Detailed comparison between the two cases (with and without adhesion) is shown later in Figure 7.

Now, we subject beds of even more cohesive particles (\( \mathbf{Bo} = 20 \)) to mechanical vibration of a single-frequency sinusoidal oscillation in the direction of gravity. When the vibration intensity is strong enough (when \( \Gamma > 0 \)), even these highly cohesive beds get fluidized in the sense that 1D-TW reappears (Figure 3). At a fixed gas flow rate (we assume it to be time-independent; see the discussion in section 2.3), the wavelength apparently increases with \( \Gamma \); however, the wave speed remains almost the same (see Figure 3). We define the critical Bond number \( \mathbf{Bo}_c \) as the maximum value of \( \mathbf{Bo} \) for which the bed is fluidizable (generating 1D-TW) at given set of oscillation parameters, and compute it as functions of \( \Gamma \) and \( f \), using a bisection-type search along the \( \mathbf{Bo} \)-axis. We find that \( \mathbf{Bo}_c \) increases almost linearly with \( \Gamma \) (Figure 4), but it is only weakly dependent on \( f \) (Figure 5). The frequency range shown in this figure corresponds to the usual operation range of 10–100 Hz, for the particle size of \( d_p = 50 \mu m \). In this range of frequencies, \( \mathbf{Bo}_c \) is virtually independent of the frequency. As the gas flow rate increases, \( \mathbf{Bo}_c \) slightly increases at fixed values of \( \Gamma \) and \( f \) (see the cases for \( \Gamma = 3 \) and 6 in Figure 4).
3.3. The Role of Vibration: Direct Impact versus Pressure Pulsation. In this section, we discuss the manner in which the vibrating base plate, or the distributor, interacts with the bed material.

In the absence of gas, the kinetic energy of individual particles in vibrated beds is obtained only from direct impact with the plate and is dissipated through interaction between particles. In shallow beds \((H_0 < 20d_p)\), the fluctuating kinetic energy (granular temperature) dissipates so quickly through collisions that the entire bed can be well-approximated by one solid body. The dynamics of the center of mass of such a bed can be described by that of a single perfectly inelastic ball on a vibrating plate. Vibrated shallow beds in a vacuum undergo period doubling bifurcations as \(\Gamma\) is varied. Such layers of a large aspect ratio form various spatiotemporal standing wave patterns. However, the temporal dynamics of vibrated deep beds (with a depth of \(20d_p\) or more) under vacuum are not commensurate with the oscillation frequency and they exhibit more complicated nonperiodic behavior.

When the gas-phase effects are taken into account, the gas drag causes the dynamics of vibrated deep beds to deviate from those under vacuum, and the deviation is more pronounced for smaller particles (because the gas drag is larger). For the deep beds of fine powders considered here, the presence of a gas phase modifies the bed dynamics so that the temporal dynamics become periodic. In the absence of a net flow \((U_s = 0)\) the bed lifts off from the plate only slightly (even smaller than the particle size) during a fraction of a cycle, and the bed impacts the plate later in the same cycle (dot-dashed lines in Figure 6). This periodic behavior has the same periodicity as the plate oscillation. Even for higher values of \(\Gamma\), the period doubling phenomenon, which occurs in vibrated shallow layers under vacuum, is not observed. Note that, in the early phase of the oscillation cycle shown in Figure 6 \((0 < t/T < 0.25)\), the plate moves downward (as does the base plate), and yet the bottom surface of the bed is approaching the distributor plate, i.e., the bed is descending faster than the plate. The bed hits the plate, remains in contact for a duration of time, and then detaches from it.

When the gas flow is turned on and its rate is increased, the velocity of the bed (relative to the base plate) during its short flight increases and the bed lifts off further from the plate (Figure 6). The upward gas flow resists the downward motion of the bed; hence, not only the duration of direct impact but also its magnitude (strength) gradually decreases (see the top panel in Figure 7) as the gas flow rate increases. When the adhesion at the base plate is taken into account (dashed lines in Figure 7), the plate experiences some force, even when the bulk of the bed is in flight (for \(t/T\) values less than \(\approx 0.2\) or greater than \(\approx 0.6\)), because there are a small number of particles stuck to the plate. Other than this, compared to the case when the
adhesion is neglected (solid lines), no difference is observed. The bed eventually hardly touches the plate at some gas flow rate, above which direct impact remains minimal. It will be shown in the next section that the minimum gas flow rate at which the direct impact virtually vanishes is, for all practical purposes, the same as the minimum fluidization velocity in vibrated beds.

### 3.4. Pressure Drop in Vibrated Beds

The pressure drop across vibrated beds oscillates with the same periodicity as the plate oscillation (see the bottom panel in Figure 7). The pressure drop increases (decreases) when the base plate moves down (up) (see Figure 6 and the bottom panel in Figure 7, for \( \frac{t}{T} \approx -0.5 \)). At first glance, this seems to be counter-intuitive, because the pressure drop increases when the plate is “moving away”. However, note that the change in the pressure drop is determined by the change in the gap between the bed and the base plate (i.e., the relative motion, with respect to the plate), not by the absolute motion of the base plate in the laboratory frame. As noted in the previous section, the bed approaches the plate during the phase of the oscillation cycle when the plate is moving down from its mean position.

As soon as impact occurs, the pressure drop begins to decrease rapidly, even below zero. While the bed is moving away from the plate after the takeoff (see \( \frac{t}{T} > -0.6 \) in Figure 6), the pressure drop continues to decrease. During this time, a region of lower pressure is being created in the gap between the bed and the plate (see the bottom panel in Figure 7). Our results are generally consistent with the experimental measurements by Thomas et al.,

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When the vertical component of the stress at the plate is combined with the pressure drop for each case, the resulting curves for different gas flow rates virtually coincide with the same sinusoidal curve (see the inset in the bottom panel of Figure 7). This can be understood from the following force balance relation in the direction of gravity:

\[
\frac{\sigma_{\text{plate}}}{\sum mg} + \frac{\Delta p}{\sum mg} = \frac{g_{\text{eff}}}{g}
\]

(17)

where \( \sum mg \) is the weight of the bed per unit cross-sectional area, and the restoring force due to the softness of the particles (i.e., the spring stiffness) is neglected. The above relation holds at every moment during a cycle, and the asymptotic common sinusoidal curve in the inset is identified to be \( 1 + \Gamma \sin(2\pi ft) \).

It corresponds to the effective gravity in the vibrating plate frame \( g_{\text{eff}} \); see the right-hand side of eq (17). Using the fact that the pressure drop across the bed is limited by \( (1 + \Gamma)\sum mg \) during a cycle, we can examine the validity of the incompressibility assumption for the gas phase that we use. By making use of the equation of state for an ideal gas, one can show that the ratio of the change in gas-phase density \( \Delta \rho / \rho_{\text{ref}} \) to its reference value \( \rho_{\text{ref}} \) satisfies the relation \( \Delta p / \rho_{\text{ref}} \approx \Delta p / \rho_{\text{atm}} = (1 + \Gamma)\sum mg / \rho_{\text{atm}} \), where \( \rho_{\text{atm}} \) is the atmospheric pressure.

For the beds of fine powders considered in our study, \( \sum mg / \rho_{\text{atm}} \approx (10^{-5}) \); hence, \( \Delta p / \rho_{\text{atm}} \ll 1 \) and the assumption of the incompressibility is valid at every moment during a cycle.

To test the sensitivity of the results to the spring stiffness, we repeated the calculations shown in Figure 7 with an order-of-magnitude-larger value of \( k_n \). The results are presented in Figure 8. The detailed behavior during a cycle is surely dependent on the value of the spring stiffness; the beds of softer particles in Figure 7 noticeably further compress and expand during the impact, and the pressure drop continues to decrease to even below zero during the impact. This effect diminishes when a larger value is used for \( k_n \); despite the quantitative changes that are readily apparent, the results shown in Figures 7 and 8 are qualitatively similar. Importantly, in a fully fluidized state, pressure pulsation is the only relevant mechanism in both cases, and the value of \( k_n \) becomes irrelevant.

Viewing the pressure pulsation at the bottom plate as a forcing set up by the plate, one can inquire about the speed at which this pulsation propagates upward through the bed. Because we have considered the gas phase to be incompressible, the pulse propagates almost instantaneously. Thus, at each time instant, the gas pressure decreases essentially monotonically as one moves up through the bed (except for the small periodic variation associated with the voidage waves). If one allows the

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**Figure 7.** (Top) The normal stress tensor on the base plate, scaled by the bed weight per unit cross-sectional area (\( \sum mg \)), is shown during a cycle for different superficial gas flow rates: \( Bo = 20, \Gamma = 3.0, f = 0.16, \) and \( k_n = 2.0 \times 10^5 \). Solid lines are obtained by neglecting adhesion at the base plate (\( Bo_n \), the \( Bo \) at the plate, is equal to zero). Dashed lines are obtained when the adhesion is taken into account (\( Bo_n = 40 \), for two cases \( U_i = 0.0082 \) and 0.024), which are almost the same as the nonadhesive cases. A dot—dashed line, which represents the plate trajectory in an arbitrary unit, is drawn to represent the phase angle during a cycle. (Bottom) The scaled pressure drop for the same cases as in the top panel are shown. Inset shows the sum of the pressure drop and the normal stress at the base plate for three cases \( (U_i = 0.0, 0.016, \) and 0.033), all of which virtually coincide with the reduced effective gravity \( g_{\text{eff}} \) in the vibrating frame (sinusoidal curve; \( 1 + \Gamma \sin(2\pi ft) \)). Small excess amounts, appearing as small peaks on top of a sinusoidal curve, arise from the transient restoring force of the soft particles at the impact, which gradually disappears as the plate impacts the bed more gently with increasing \( U_i \) (there are no apparent additional peaks for \( U_i = 0.033 \)).
gas to be compressible, then the pulsation travels at the speed of sound; sound speed through gas-fluidized beds is considerably smaller than that through a column of gas. If the time required for the propagation of the pulsation through the bed is commensurate with the period of the plate oscillation, resonance can set in; however, such resonance is suppressed in the present study because we have treated the gas as incompressible. In any case, in the relatively shallow beds that we consider, resonance is not expected to be a significant effect.

The pressure drop averaged throughout a cycle, as a function of gas flow rate, manifests a linear increase up to the constant plateau region (Figure 9), which is qualitatively the same as in conventional fluidized beds. The minimum velocity when the direct impact virtually does not occur is essentially the same as the minimum fluidization velocity in vibrated beds; only the pressure pulsation is a relevant mechanism for a fluidized state of vibrated beds. Note that the minimum fluidization velocities for the vibrated beds of cohesive particles are larger than that for a nonvibrated bed of noncohesive particles of the same size. This can be interpreted as the increment in effective particle size, which is understandable, because the fluidized entities in vibrated beds of cohesive particles are agglomerates, not individual particles. The average volume fraction at the minimum fluidization is smaller (e.g., $\phi_{\text{min}} = 0.631$ for the case of $Bo = 20$), compared to what is observed in a bed of noncohesive particles ($\phi_{\text{min}} = 0.652$); cohesive beds have a tendency to pack more loosely. In the fully fluidized state, the pressure drop exhibits a plateau, which approximately equals the weight of the bed per unit cross-sectional area (see Figure 9), as in nonvibrated beds. There is no clear consensus on this issue in experimental studies. Tasirin and Anuar\textsuperscript{29} reported the pressure drop increases as the vibration intensity $\Gamma$ increases, Erdész and Mujumdar\textsuperscript{28} observed the opposite trend, and Marring et al.\textsuperscript{7} observed that the constant plateau pressure drop was equal to the bed weight per unit cross-sectional area in high gas flow rates. Only the latter is consistent with our results, which can be explained by the simple force balance argument in eq 17, accounting for the fact that the direct impact is negligibly small in a fluidized state.

3.3. Breakup of Cohesive Assembly by Pressure Pulsation.

We seek to understand how vibration facilitates the breakup of cohesive assemblies into agglomerates and maintains the propagation of the wave in a fluidized state. We analyze the profile of continuum variables, including the granular temperature $T$ and the solid-phase stress tensor $\sigma$ (or the solid phase pressure) across the traveling wave during a cycle. Because the waves in a bed of finite depth that we have considered thus far are not perfectly periodic, we consider an alternate, idealized geometry, where a wave is fully developed in a small periodic box (in all of three directions) of height $L$ that is commensurate with the wavelength obtained in the vibrated fluidized bed simulations described above. Note that the direct impact does not have an important role for a fully fluidized bed (see section 3.3; only the pressure pulsation has an important role), and the vibrating plate does not have to be considered in such a case. For a fully fluidized state, the weight of the bed per unit cross sectional area is supported by the pressure drop:

$$p|_{z=0} - p|_{z=L} = \rho p \phi_{\text{avg}} L \sigma_{\phi_{\text{avg}}}.$$

where $\phi_{\text{avg}}$ is the average volume fraction.

Comparison between fully fluidized states of cohesive beds and noncohesive beds on microscopic level reveals that cohesive particles form strings of particles or agglomerates while they rain down through void regions, whereas noncohesive particles come down individually (see Figure 10; full animations are available from http://multiphase.princeton.edu/ICE_2005). As one can readily see, this effect arises from the attractive force between cohesive particles, which can be well-characterized by tensile stress on a continuum level. We compute the stress tensor, which consists of a kinetic or dynamic part and a virial or static part, using the following microscopic relation:\textsuperscript{66}

$$\sigma = \frac{1}{V} \sum_{i} \left[ \mathbf{v}_{i} \otimes \mathbf{v}_{i} - \sum_{c \in V} \mathbf{f}_{c} \otimes \mathbf{l}_{c} \right]$$

Figure 9. Average pressure drop during a cycle, scaled by the bed weight per unit cross-sectional area, obtained for different cases of vibrated fluidized beds, as a function of $U_s$. The pressure drop in conventional fluidized beds of noncohesive particles (diamonds) is included for comparison. Filled circles are for the case of $Bo = 20$ and $\Gamma = 3.0$, but the value of spring stiffness that was used ($k_s = 2.0 \times 10^6$) is 10 times larger, compared to all the other cases ($k_s = 2.0 \times 10^5$).

Mawatari et al.\textsuperscript{9} and Nam et al.\textsuperscript{11} observed that the constant plateau pressure drop was equal to the bed weight per unit cross-sectional area in high gas flow rates. Only the latter is consistent with our results, which can be explained by the simple force balance argument in eq 17, accounting for the fact that the direct impact is negligibly small in a fluidized state.
where $\bar{v}_i$ is the fluctuating velocity of the $i$th particle ($\bar{v}_i = v_i - \langle v_i \rangle$), $\otimes$ is the dyadic tensor product, $f_c$ is the interacting force between contacting particles 1 and 2, and $L_c$ is the displacement vector between the centers of particles under consideration ($L_c = r_1 - r_2$). The second term is summed over all the contacts in the averaging volume $V$.

Now we consider the continuum level interpretation of vibrated fluidized beds of highly cohesive particles in a fully periodic box during an oscillation cycle (Figure 11). During a cycle, the wave oscillates up and down with a net upward motion, as can be seen in the cases shown in Figure 3. We compute the solid-phase continuum variables, including the pressure and the trace of the stress tensor per dimension ($\text{Tr}(\sigma)/D$). We compute them in the co-travelling frame (with the wave’s net motion), averaged over 100 cycles, for the purpose of variance reduction. As the pressure pulsation cyclically varies during a cycle, so do both the granular temperature and the stress tensor.

Figure 11 shows the variation of solid-phase continuum variables at six different times during an oscillation cycle. The
inset in each panel shows the position of a hypothetical, oscillating base plate at the instant the profiles of the continuum variables are shown. The plate trajectory sketched in the insets can be compared to that shown in the top panel of Figure 7. The corresponding pressure drop across the periodic domain can be determined from the cyclically varying pressure drop profile in the bottom panel of Figure 7, which is almost out of phase with the plate position. In Figure 11a, the value of $\frac{\partial \phi}{\partial t}$ and the scaled pressure drop are almost unity; they are larger in Figure 11b, attain their largest values near the instant shown in Figure 11c, relax back toward unity in Figure 11d, and then they decrease in Figure 11e, until they reach the minimum values in Figure 11f.

When the scaled pressure drop is almost unity (Figure 11a), the solid-phase pressure is negligibly small throughout the bed. As the pressure drop increases until it reaches its maximum value (Figures 11b and c), the granular temperature and compressive stress in the lower plug and tensile stress in the upper plug also increase significantly, until they reach their maximum magnitudes. The large granular temperature occurs at the lower plug, where the particles or agglomerates are accumulated. The phenomenon results from vigorous collisions among the particles or agglomerates that are raming down and the lower plug. Note that the volume fraction in this region is still low (the value of $\phi$ is less than $\sim 0.3$). A rough estimate for the magnitude of the stress using the values at the maximum temperature region ($\sim nT = 6(\pi d_{p}^{2})\phi T \approx 0.2$), where the most vigorous collisions occur, shows that it is still much smaller than the tensile strength of the cohesive assembly ($\sim 1$) that is estimated below; here, $n$ is the number density. In order for these vigorous collisions to contribute to the breakup of the cohesive assembly, the magnitude of the stress formed by the collisions must be comparable or larger than the tensile strength of the material. However, the stresses formed by the collisions are not strong enough to break up the assembly and are irrelevant for the assembly breakup. Rather, it is the increased tensile stress in the upper plug that breaks up the assembly into agglomerates and maintains the wave propagation. Particles in the upper plug can split off from the assembly and rain down, because the increased tensile stress becomes large enough to reach the tensile strength of the cohesive assembly. It occurs at a time near that shown in Figure 11c.

The magnitude of the tensile stress cannot exceed the strength of the assembly; therefore, we estimate the strength of the fluidized bed by measuring the maximum tensile stress. We compare such obtained tensile strength of the fluidized bed with the prediction of Rumpf’s model,\(^{57}\)

$$\sigma_t = \left(1 - \frac{\epsilon}{\kappa}\right)\left(\frac{F_1}{d_{p}}\right)$$

where $\sigma_t$ is the tensile strength, $\epsilon$ the porosity ($\epsilon = 1 - \phi$), $k$ the coordination number, and $F_1$ the cohesive contact force. We find that our measurement ($\sigma_t \approx 0.8$) is approximately an order of magnitude smaller than the prediction ($\sigma_t \approx 6$). This discrepancy is understandable, because the cohesive assembly in a fluidized bed breaks up through the weakest linkage, as opposed to all directions, as is assumed in Rumpf’s model.

Later in the cycle, when pressure drop decreases and approaches the minimum value (Figure 11d–f), the particle phase pressure is generally small in the bed. When the pressure drop is negative, a relatively small compressive stress amasses at the bottom of the upper plug as the gas in the void region pushes up; this is obviously irrelevant for the breakup.

4. Summary

We have used a particle-dynamics-based model for vibrated gas-fluidized beds of fine powders to study how the vibration facilitates the fluidization of beds of cohesive powders. We have demonstrated that, as the gas flow rate increases, the direct impact from the plate decreases and the pressure pulsation becomes more dominant. In a fluidized state, the latter is shown to be virtually the only relevant mechanism, and the pressure drop follows a simple sinusoidal curve during a cycle (see Figures 7 and 8), which corresponds to the weight of the bed per unit cross-sectional area in the vibrated frame. As a consequence, the pressure drop averaged over a cycle in the fluidized state is simply the offset of this sinusoidal curve, which equals the weight of the bed per unit cross-sectional area, as in nonvibrated beds. This relation can be readily understood by a simple force balance argument (see eq 17). In a bubbling bed (which appears as 1D-TW in our study), it is during the transient time interval with large enough pressure pulsation when the increased tensile stress breaks up the cohesive assembly into agglomerates (see Figure 11).

Note that the compressibility of the gas phase was ignored in the present study; it would be interesting to investigate the resonance effects that may arise in vibrated beds by allowing for gas compressibility. It was also assumed that the gas superficial velocity is independent of time throughout the oscillation cycle, and this corresponds to the limit of very large resistance for gas flow through the distributor; it would also be interesting to examine the case of finite distributor resistance, where the superficial gas velocity can be expected to vary cyclically with plate vibration.

While the present study has yielded a physical understanding of the pressure fluctuations induced by the plate and the tensile stress in the particle assembly, an analytical relation between vibration parameters and the agglomerate size is still elusive. Furthermore, we have considered only beds of narrow cross-sectional areas, and we have assumed the volume-averaged gas phase to be one-dimensional. Consequently, some generic behavior in real vibrated gas-fluidized beds, such as the horizontal sloshing motion of the particles, meandering gas flows around agglomerates, and more complicated bubble dynamics, were not allowed to occur. By avoiding such complexity, we were able to bring out certain basic physics of the bed dynamics. Future studies should investigate the dynamics of higher-dimensional beds; however, simulation of fluidized beds of realistic industrial scales, using the current approach, is not yet feasible.

Acknowledgment

This work was funded by The New Jersey Commission on Science and Technology, Merck & Co., Inc., and an NSF/ITR grant. We are delighted to contribute this article to the special issue honoring W. B. Russel. Having him as a colleague and a friend has been a pleasure and a privilege for both S.S. and I.G.K. We salute his accomplishments and look forward to many more years of work and fun together.

Nomenclature

$A$ = Hamaker constant
$B_0$ = cohesive Bond number between particles
$B_{0p}$ = cohesive Bond number between particles and the base plate
$C_D$ = drag coefficient
$\Delta p$ = pressure drop across the bed
\[ \Delta x \] = grid spacing for coarse-grained variables

\[ d_p \] = particle diameter

\[ e \] = normal coefficient of restitution

\[ f \] = vibration frequency

\[ F_{\text{cont}} \] = interaction force due to contact

\[ F_c \] = cohesive force due to van der Waals force

\[ g_{\text{eff}} \] = effective gravitational acceleration

\[ h \] = halo function

\[ k \] = coordination number

\[ k_n \] = spring stiffness in the normal direction

\[ k_t \] = spring stiffness in the tangential direction

\[ m_p \] = mass of individual particle

\[ n \] = solid-phase number density

\[ p_{\text{am}} \] = atmospheric pressure

\[ p \] = gas-pressures

\[ Re_T \] = Reynolds number based on particle size

\[ r \] = particle radius

\[ s \] = intersurface distance

\[ S_t \] = Stokes number

\[ T \] = granular temperature

\[ \sum m g \] = bed weight per unit cross-sectional area

\[ u_e \] = volume-averaged gas-phase velocity

\[ u_i \] = volume-averaged solid-phase velocity

\[ U_i \] = superficial gas-flow velocity

\[ U_{\text{min}} \] = minimum fluidization velocity

\[ v_n \] = relative velocity in the normal direction

\[ v_t \] = relative velocity in the tangential direction

\[ v_p \] = velocity of an individual particle

\[ V_p \] = volume of an individual particle

**Greek Symbols**

\[ \beta \] = interphase momentum transfer

\[ \delta \] = minimum separation distance for cohesion

\[ \Delta \eta \] = tangential displacement from initial contact

\[ \eta \] = amount of overlap in the normal direction

\[ \epsilon \] = porosity (gas-phase volume fraction)

\[ \phi \] = solid-phase volume fraction

\[ \gamma_n \] = damping coefficient for dashpot

\[ \Gamma \] = vibration intensity

\[ \mu \] = coefficient of friction

\[ \mu_g \] = gas-phase viscosity

\[ \nu_p \] = Poisson’s ratio

\[ \rho_g \] = gas-phase mass density

\[ \rho_s \] = solid-phase mass density

\[ \sigma \] = solid-phase stress tensor

\[ \sigma_t \] = tensile strength of the material

\[ \sigma_{\text{plate}} \] = vertical stress on the plate

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*Received for review October 5, 2005 Revised manuscript received January 10, 2006 Accepted January 11, 2006*